

On the modeling and simulation of the nonlinear dynamic response of NEMS via a couple of nonlocal strain gradient theory and classical beam theory

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Abstract. In the present research, the dynamic characteristics of the nanoscale tubes and pipes with nonuniform cross-sections are examined. The aforementioned nanostructures are made by imperfect axially functionally graded materials (AFGM) that compose ceramic and metal phases along the tube length direction, involving the porous voids. To this purpose, the Hamilton principle is implemented to obtaining the governing equation and related boundary conditions using classical beam theory coupled to the nonlinear Von-Kármán theory. In order to apply the size impact, the nonlocal strain gradient theory is considered that both hardening and softening parameters are involved. Also, iteration techniques, including the generalized differential quadrature method (GDQM), are used to solve linear and nonlinear derived partial differential equations (PDE). Finally, the obtained results are explained in detail to investigate the impact of nonlinear amplitude, nonlocal and strain gradient parameter, porosity parameter, etc., for both clamped and simply-supported types of boundary conditions, which are helpful to design the nanoelectromechanical structures (NEMS).

Keywords: axially functionally graded tube; gradient strain theory; imperfect; nonlinear vibration; non-uniform; porous; tapered tube; truncated conical tube

1. Introduction

Nanostructures have played an important role in the advancement of sciences, engineering, and new nanotechnologies. Nanomaterials are gaining popularity owing to their unique features that influence biological, chemical, optoelectrical, electrical, and physical properties. The focus on nanostructures is one of the most important subjects that attracts many academics to develop these studies concerning the behavior of nanostructures. Many scientists have focused their efforts in recent years on creating nanoscience in many areas, particularly nanomechanics, such as nano-scale beams, microbeams, nanoplates, and many other structures in nano and micro sizes (Hamidi *et al.* 2015, Allahkarami *et al.* 2017, Ehyaei *et al.* 2017, Akbas 2018a, b, Arefi and Zenkour 2018, Aydogdu *et al.* 2018, Bensaid *et al.* 2018, Navi *et al.* 2019, Ebrahimi *et al.* 2020, Gafour *et al.* 2020, Matouk *et al.* 2020, Ni *et al.* 2020, Zhao *et al.* 2020, 2021, Guo *et al.* 2021, Hu *et al.* 2021, Li *et al.* 2021, Liu *et al.* 2021a, b, c, Lu *et al.* 2021, Ni *et al.* 2021, Wang *et al.* 2021, Zhang *et al.* 2021a, b, Zhong *et al.* 2021).

Vibration of nano beams and tubes are of great importance in the past years in the literature. Besseghier *et al.* (2015) studied the nonlinear vibration of a zigzag shaped nanotube using Winkler-type model to consider the interaction of the zigzag nanotube with the surrounding

elastic medium. Alibeigloo and Emtehani (2015) used the three dimensional theory of elasticity to study the vibration of carbon nanotubes reinforced composite (CNTRC) plates. Mirzaei and Kiani (2016) investigated the large amplitude free vibrations of a sandwich beam with stiff core and carbon nanotube (CNT)-reinforced face sheets. Wang *et al.* (2016) introduced the multi-term Kantorovich–Galerkin method to analyze the buckling and vibration of thin carbon nanotube-reinforced composite (CNTRC) plates with the classical plate theory. Lu *et al.* (2017) studied the size-dependent sinusoidal shear deformation nanobeam model to investigate the free vibration of nanobeams based on the nonlocal strain gradient theory. Ghadiri and Shafiei (2016) studied the small scale effect on the nonlinear bending vibration of a rotating cantilever and propped cantilever nanobeam. Investigation on free vibration of nanobeams based on Eringen nonlocal elasticity theory and Timoshenko beam theory is presented by Ebrahimi and Nasirzadeh (2015).

Functionally graded materials (FGM) have been used and studied for many applications such as sensors (Mueller *et al.* 2003), biomedical applications (Pompe *et al.* 2003), thermal barrier coatings (Lee *et al.* 1996), orthopedic applications (Bekkaye *et al.* 2020, Zine *et al.* 2020, Arshid *et al.* 2021, Bellifa *et al.* 2021, Guellil *et al.* 2021, Hadji *et al.* 2021, Kumar *et al.* 2021, Tahir *et al.* 2021a, b), automotive applications (Srinivas *et al.* 2019), biological materials (Liu *et al.* 2017) and etc (Bai *et al.* 2020, Li *et al.* 2020a, b, Al-Furjan *et al.* 2021a, b, c, d, Dai *et al.* 2021a, b, Guo *et al.* 2021, Habibi *et al.* 2021, Huo *et al.* 2021, Shi *et al.* 2020, Wang *et al.* 2020, Zhou *et al.* 2020, Najaafi *et al.* 2021, Peng *et al.* 2021, Shao *et al.* 2021, Wu and Habibi

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2021, Zhang *et al.* 2021). These various applications made the researchers to study the vibrational behavior of nanosystems which are made by FGMs. Zhang *et al.* (2015) studied the frequencies of functionally graded carbon nanotube-reinforced composite (FG-CNTRC) skew plates and the mode shapes of them with the transverse shear deformation. Şimşek (2016) used nonlocal size-dependent nonlinear vibration of functionally graded nanobeam. The thermo-mechanical vibration behavior of two dimensional functionally graded (2D-FG) porous nanobeam is studied by Mirjavadi *et al.* (2017). Shafiei *et al.* (2016) studied nonlinear vibration of rotating Euler–Bernoulli porous nanobeam considering the small-scale effect. Mirjavadi *et al.* (2017) considered the thermal effect in their investigation on the buckling and free vibration of axially functionally graded nanobeam. Mohammadi *et al.* (2019) considered mechanical an electrical loads on nonlinear vibration porous functionally graded Euler-Bernoulli nanobeam. In another study, Tong *et al.* (2020) studied the stability of functionally graded nanotube with cantilever boundary condition, also, wave propagation in embedded functionally graded nanotubes conveying fluid is studied (Filiz and Aydogdu 2015).

It is apparent from the numerous experiments given that various parameter effects affect the nonlinear response of FG tubes. Despite the availability of many references on nonlinear responses of nanostructures, no reference has been given to explore the nonlinear behavior of an AFG tapered and imperfect nanotube with nonlinear nonlocal boundary conditions. As a result, more research into the nonlinear vibrational of functionally graded truncated conical nanosized tubes with porosity is required. For the first time, this article examines the nonlinear analysis of a nonuniform porosity-dependent nanotube constructed of metal and ceramic graded material, taking into account von-nonlinear Kármán's strain. The nonlinear equations and nonlinear nonlocal boundary conditions are obtained using the Hamilton principle, and they are solved using GDQ and the iteration technique for the classical Euler-Bernoulli beam theory.

2. Conceptualization and formulation

2.1 Tubes structures

Along Fig. 1, in the axial direction of the tube, a combination of a metal (at $x=0$) and a ceramic (at $x=L$) phase produces a tapered AFG-nanotube with lengths 'L', 'R_i', and 'R_o', as well as inner and outer radiuses. The material variation leads to a variation in all the nanotube's mechanical properties following the relation in Eq. (1), where "n_R" indicates the FG power index (Shafiei and Kazemi 2017).

$$F(r) = F_m + (F_c - F_m) \left(\frac{x}{L}\right)^n \quad (1)$$

The ()_c and ()_m indices represent ceramic and metal, respectively, and 'r' is the distance from the center of the FG tube. The linear function of radius changes the radius of

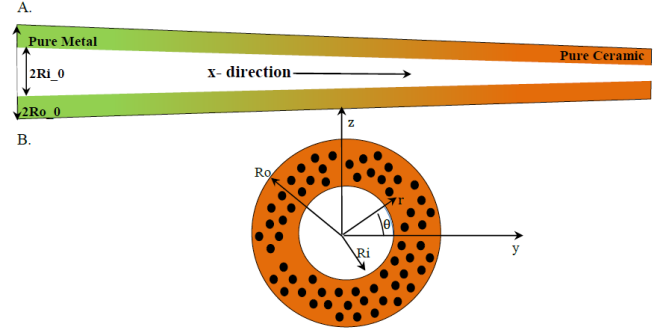


Fig. 1 The non-uniform cross-section and material change of the Functionally graded porous nano-tube throughout its length

Table 1 The mechanical characteristics of several ceramic and metal phases, involving Young's modulus, Poisson ratio, and mass density (Reddy and Chin 1998)

Properties	SUS304	Si ₃ N ₄
E (Pa)	2.0779e+11	3.22269e+11
ν	0.3177	0.24
ρ (Kg/m ³)	8166	2370

the non-uniform cross-section tube throughout its length; the linear function of radius is:

$$r = R_0 \left(1 + \beta_R \left(\frac{x}{L}\right)\right) \quad (2)$$

R₀ indicates the radius in the left side (at $x = 0$) of the tube, and β_R is the rate of cross-section changes. Both inner and outer radius are applied by this function. The equations for Young's modulus (E), Poisson's ratio (ν), and mass density (ρ) equations of the FG nanotube can be expressed as:

$$E(x) = \left[E_m + (E_c - E_m) \left(\frac{x}{L}\right)^n \right] \left(1 - \alpha \cos \left(\pi \frac{r - R_i}{R_0 - R_i} \right) \right) \quad (3a)$$

$$\nu(x) = \left[\nu_m + (\nu_c - \nu_m) \left(\frac{x}{L}\right)^n \right] \left(1 - \alpha \cos \left(\pi \frac{r - R_i}{R_0 - R_i} \right) \right) \quad (3b)$$

$$\rho(x) = \left[\rho_m + (\rho_c - \rho_m) \left(\frac{x}{L}\right)^n \right] \left(1 - \alpha \cos \left(\pi \frac{r - R_i}{R_0 - R_i} \right) \right) \quad (3c)$$

α (0 ≤ α << 1) is the volume fraction of porosity (Shafiei and Kazemi 2017). The mechanical properties of different ceramics and metals are shown in Table 1. The coordinate system is considered as cylindrical O (x, r, θ). In this coordinate system, x, r and θ define the length, radial, and circumferential directions, respectively. The coordinate system to refer the nanotube is rectangular O (x, y, z), where u₁, u₂, and u₃ denote the appropriate displacement components are as follows:

$$y, z = r (\cos(\theta), \sin(\theta)); \quad y^2 + z^2 = r^2 \quad (4)$$

2.2 Mathematical modeling

The classical beam model is used to calculate the displacement of the nanotube's point, as shown below:

$$\begin{aligned} u_1(x, z, t) &= z\varphi + u(x, t) \\ u_2 &= 0 \\ u_3(x, t) &= w(x, t) \end{aligned} \quad (5)$$

The transverse displacement and rotation angle around the y axis are represented by $w(x)$ and ' $\varphi = -\partial w / \partial x$ ', respectively. Also, where u_1 , u_2 and u_3 are the displacement field components along the x , y , and z axes. The nonlinear strains are defined as follows using the nonlinear Von-Kármán theory linked with the classical and first-order shear deformation beam theory:

$$\varepsilon_{xx} = \frac{\partial u_1}{\partial x} = \frac{\partial u}{\partial x} - z \frac{\partial^2 w}{\partial x^2} + \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 \quad (6a)$$

$$\varepsilon_{xy} = \varepsilon_{xz} = 0 \quad (6b)$$

stresses are defined as:

$$\sigma_{xx} = E \varepsilon_{xx} \quad (7a)$$

$$\sigma_{xy} = \sigma_{xz} = 0 \quad (7b)$$

The Hamiltonian principle is used to derive the governing equations as well as the boundary conditions equations (Hou *et al.* 2021, Huang *et al.* 2021a, b, c, Jiao *et al.* 2021, Liu *et al.* 2021 a, b, c, Ma *et al.* 2021, Moradi *et al.* 2021, Xu *et al.* 2021, Zhao *et al.* 2021, Yu *et al.* 2022). It is explained as follows:

$$\int_{t_1}^{t_2} \delta H dt = \int_{t_1}^{t_2} (\delta K - \delta S + \delta V) dt = 0 \quad (8)$$

where K , S , and V denote, respectively, kinetic energy, strain energy, and external work. Transversally, the displacement is believed to be considerable, whereas the rotation is believed to be minor. The following equation is used to determine the nanotube's virtual strain energy:

$$\delta S = \iiint \delta s dv = \int_0^L \left[A_{11} \frac{\partial u}{\partial x} \delta \left(\frac{\partial u}{\partial x} \right) + D_{11} \frac{\partial^2 w}{\partial x^2} \delta \left(\frac{\partial^2 w}{\partial x^2} \right) + \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 \delta \left(\frac{\partial w}{\partial x} \right) + A_{11} \left(\frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^3 \delta \left(\frac{\partial w}{\partial x} \right) + \frac{\partial u}{\partial x} \frac{\partial w}{\partial x} \delta \left(\frac{\partial w}{\partial x} \right) \right] dx \quad (9)$$

where

$$(A_{11}(x), D_{11}(x)) = \iint_A E(x, r)(1, z^2) dA \quad (10)$$

The kinetic energy is derived as:

$$\begin{aligned} \delta K &= \delta \iiint k dv = \frac{1}{2} \delta \int_0^L \iint_A \rho(x, r) \left[\left(\frac{\partial u_x}{\partial t} \right)^2 + \left(\frac{\partial u_y}{\partial t} \right)^2 + \left(\frac{\partial u_z}{\partial t} \right)^2 \right] dA dx \\ &= \int_0^L \left\{ m_0 \left[\frac{\partial u}{\partial t} \delta \left(\frac{\partial u}{\partial t} \right) + \frac{\partial w}{\partial t} \delta \left(\frac{\partial w}{\partial t} \right) \right] + m_2 \frac{\partial^2 w}{\partial x \partial t} \delta \left(\frac{\partial^2 w}{\partial x \partial t} \right) \right\} dx \end{aligned} \quad (11)$$

where

$$(m_0(x), m_2(x)) = \iint_A \rho(x, r)(1, z^2) dA \quad (12)$$

External work (V) has been set to zero.

$$\delta V = 0 \quad (13)$$

The following Euler–Lagrange equation has been found by substituting Eqs. ((9), (11) and (13) into Eq. (8) and assuming that δu , δw and $\delta \psi$ are all zero:

$$\delta u : -\frac{\partial}{\partial x} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) = m_0 \frac{\partial^2 u}{\partial t^2} \quad (14a)$$

$$\delta w : \frac{\partial^2}{\partial x^2} \left(D_{11} \frac{\partial^2 w}{\partial x^2} \right) + \frac{\partial}{\partial x} \left[A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right] \frac{\partial w}{\partial x} = m_0 \frac{\partial^2 w}{\partial t^2} - m_2 \frac{\partial^4 w}{\partial x^2 \partial t^2} \quad (14b)$$

Furthermore, the following are the boundary conditions:

$$u = 0 \quad \text{or} \quad N = 0 \quad (15a)$$

$$\partial w / \partial x = 0 \quad \text{or} \quad M = 0 \quad (15b)$$

$$w = 0 \quad \text{or} \quad \partial M / \partial x = 0 \quad (15c)$$

where

$$N = A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \quad (16a)$$

$$M = D_{11} \frac{\partial^2 w}{\partial x^2} \quad (16b)$$

2.3 Nonlocal strain gradient theory

Lim *et al.* (2015) developed nonlocal strain gradient theory by combining Eringen (1983) nonlocal stress theory with Aifantis (1992) and Mindlin (1965) strain gradient theory. The total stress tensor t_{xx} is calculated using the nonlocal strain gradient theory as follows:

$$\left[1 - (ea)^2 \nabla^2 \right] t_{xx} = E (1 - l^2 \nabla^2) \varepsilon_{xx} \quad (17)$$

where “ ea ” and “ l ” are nonlocal and strain gradient parameters. The stress resultants in Eq. (16) revised as below utilizing the nonlocal strain gradient theory:

$$N = \left(1 - l^2 \frac{\partial^2}{\partial x^2} \right) (N) + (ea)^2 \frac{\partial}{\partial x} \left(m_0 \frac{\partial^2 u}{\partial t^2} \right) \quad (18a)$$

$$M = M - l^2 \frac{\partial^2 M}{\partial x^2} + (ea)^2 \left(m_0 \frac{\partial^3 w}{\partial t^2} - m_2 \frac{\partial^4 w}{\partial x^2 \partial t^2} - \frac{\partial}{\partial x} \left((N) \frac{\partial w}{\partial x} \right) \right) \quad (18b)$$

As a consequence of the nonlocal strain gradient theory, the vibration equation for the FG nanotube is as follows.

δu :

$$\frac{\partial}{\partial x} \left(\left[l^2 \frac{\partial^2}{\partial x^2} - 1 \right] \left[\frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 + A_{11} \frac{\partial u}{\partial x} \right] \right) = -(ea)^2 \frac{\partial^2}{\partial x^2} \left(m_0 \frac{\partial^2 u}{\partial t^2} \right) + m_0 \frac{\partial^2 u}{\partial t^2} \quad (19a)$$

$$\begin{aligned}
 \delta w : & \frac{\partial^2}{\partial x^2} \left(D_{11} \frac{\partial^2 w}{\partial x^2} - l^2 \frac{\partial^2}{\partial x^2} \left(D_{11} \frac{\partial^2 w}{\partial x^2} \right) \right) - \frac{\partial}{\partial x} \left(\left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \frac{\partial w}{\partial x} \right. \\
 & \left. - l^2 \frac{\partial^2}{\partial x^2} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \right) \\
 & + (ea)^2 \frac{\partial^2}{\partial x^2} \left(\frac{\partial}{\partial x} \left(\left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 - l^2 \frac{\partial^2}{\partial x^2} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \right) \right) \frac{\partial w}{\partial x} \right) \\
 & = +m_0 \frac{\partial^2 w}{\partial t^2} - m_1 \frac{\partial^4 w}{\partial x^2 \partial t^2} - (ea)^2 \frac{\partial^2}{\partial x^2} \left(m_0 \frac{\partial^2 w}{\partial t^2} - m_1 \frac{\partial^4 w}{\partial x^2 \partial t^2} \right)
 \end{aligned} \tag{19b}$$

The nonlocal nonlinear boundary conditions are

$$\begin{aligned}
 u = 0 \text{ or } & A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 - l^2 \frac{\partial^2}{\partial x^2} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \\
 & + (ea)^2 \frac{\partial}{\partial x} \left(m_0 \frac{\partial^2 u}{\partial t^2} \right) = 0
 \end{aligned} \tag{20a}$$

$$\begin{aligned}
 w = 0 \text{ or } & \frac{\partial}{\partial x} \left(D_{11} \frac{\partial^2 w}{\partial x^2} \right) - l^2 \frac{\partial^3}{\partial x^3} \left(D_{11} \frac{\partial^2 w}{\partial x^2} \right) \\
 & + (ea)^2 \frac{\partial}{\partial x} \left(\left(m_0 \frac{\partial^2 w}{\partial t^2} - m_2 \frac{\partial^4 w}{\partial x^2 \partial t^2} \right) \frac{\partial w}{\partial x} \right) = 0
 \end{aligned} \tag{20b}$$

$$\begin{aligned}
 M = & D_{11} \frac{\partial^2 w}{\partial x^2} - l^2 \frac{\partial^2}{\partial x^2} \left(D_{11} \frac{\partial^2 w}{\partial x^2} \right) \\
 \partial w / \partial x = 0 & \left(m_0 \frac{\partial^2 w}{\partial t^2} - m_2 \frac{\partial^4 w}{\partial x^2 \partial t^2} \right) \\
 & + (ea)^2 \left(\frac{\partial}{\partial x} \left(\left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 - l^2 \frac{\partial^2}{\partial x^2} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \right) \right) \frac{\partial w}{\partial x} \right) = 0
 \end{aligned} \tag{20c}$$

The expanded equations are:

$$\begin{aligned}
 \delta w : & \frac{\partial^2}{\partial x^2} \left(D_{11} \frac{\partial^2 w}{\partial x^2} - l^2 \frac{\partial^2}{\partial x^2} \left(D_{11} \frac{\partial^2 w}{\partial x^2} \right) \right) - \frac{\partial}{\partial x} \left(\left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \frac{\partial w}{\partial x} \right. \\
 & \left. - l^2 \frac{\partial^2}{\partial x^2} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \right) \\
 & + (ea)^2 \frac{\partial^2}{\partial x^2} \left(\frac{\partial}{\partial x} \left(\left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 - l^2 \frac{\partial^2}{\partial x^2} \left(A_{11} \frac{\partial u}{\partial x} + \frac{1}{2} A_{11} \left(\frac{\partial w}{\partial x} \right)^2 \right) \right) \right) \frac{\partial w}{\partial x} \right) \\
 & = +m_0 \frac{\partial^2 w}{\partial t^2} - m_1 \frac{\partial^4 w}{\partial x^2 \partial t^2} - (ea)^2 \frac{\partial^2}{\partial x^2} \left(m_0 \frac{\partial^2 w}{\partial t^2} - m_1 \frac{\partial^4 w}{\partial x^2 \partial t^2} \right)
 \end{aligned} \tag{21a}$$

Mirjavadi *et al.* 2017a, b, c, d, Shafiei and Kazemi 2017a, b, Azimi *et al.* 2018, Shafiei and She 2018, Al-Furjan *et al.* 2020a, b, c, 2021a, b, c, d, Huang *et al.* 2021, Shariati *et al.* 2021)

$$\left. \frac{\partial^r f(x)}{\partial x^r} \right|_{x=x_p} = \sum_{j=1}^n C_{ij}^{(r)} f(x_j) \quad (22)$$

The number of grid points is specified by the letter ‘n,’ and C_{ij} as the weighting coefficient is calculated as follows:

$$C_{ij}^{(1)} = \frac{P(x_i)}{(x_i - x_j)P(x_j)}; \quad i \neq j = 1, 2, \dots, n$$

$$C_{ji}^{(1)} = - \sum_{i=1, j \neq i}^n C_{ji}^{(1)}; \quad i = j$$
(23)

where $P(x)$ is:

$$P(x_i) = \prod_{i=1, i \neq j}^n (x_j - x_i) \quad (24)$$

$C^{(r)}$ along the x-axis is calculated as follows:

$$C_{iji}^{(r)} = r \left[C_{ji}^{(r-1)} C_{ji}^{(1)} - \frac{C_{ji}^{(r-1)}}{(x_j - x_i)} \right]; \quad j \neq i = 1, 2, \dots, n$$

$$C_{ijj}^{(r)} = - \sum_{i=1, j \neq i}^n C_{ijj}^{(r)}; \quad i = j = 1, 2, \dots, n$$
(25)

The mesh point distribution is determined using the Chebyshev-Gauss-Lobatto technique as follows:

$$2x_i = \left(1 - \cos \left(\frac{(i-1)}{(N-1)} \pi \right) \right) L \quad (26)$$

Consider the FG nanotube’s nonlinear motion equations, which are a composite of the three mass matrices. The following are the linear and nonlinear stiffnesses:

$$\left\{ [K]_{Linear} + [K]_{Non-linear} - \omega^2 [M] \right\} \{ \lambda \} = 0 \quad (27)$$

Additionally, the boundary conditions are composed of three matrices. As a consequence, the following matrix represents the linear stiffness:

$$\left\{ [K] - \omega^2 [M] \right\} \{ \lambda \} = 0 \quad (28)$$

To begin, we use the extended differential quadrature method to solve the linear motion equation. After applying the weight coefficients to the linear motion equation, the following result is obtained as below:

We build the matrixes to the boundary and the governing equations while ignoring the nonlinear components. As a result, the basic linear vibration equation of the FG nano-tube is as follows:

$$\left[\begin{array}{cc} [K_{dd}] & [K_{db}] \\ [K_{bd}] & [K_{bb}] \end{array} \right] \left\{ \begin{array}{c} \{ \lambda_d \} \\ \{ \lambda_b \} \end{array} \right\} = \omega^2 \left[\begin{array}{cc} [M_{dd}] & [M_{db}] \\ [M_{bd}] & [M_{bb}] \end{array} \right] \left\{ \begin{array}{c} \{ \lambda_d \} \\ \{ \lambda_b \} \end{array} \right\} \quad (30)$$

where d and b correspond to the domain and boundary indices, respectively, and λ denotes the mode form. The nonlinear mode shapes and frequencies are obtained by coupling the nonlinear and linear stiffness matrices with the mass matrix, using the nonlinear stiffness matrix and Eq. (27) as well as the linear mode shape in Eq. (28). Iteration is then utilized to get the nonlinear frequency results by recalculating the frequency to obtain the converged nonlinear frequency (Ghadiri and Shafiei 2016, Shafiei *et al.* 2016a, b, c, d, e, f, g, 2017a, b, c, d, Shafiei and Kazemi 2017, Azimi *et al.* 2018, Hou *et al.* 2021, Huang *et al.* 2021).

$$\frac{dA_{11}(x)}{dx} \left(\sum_{s=1}^n C_{rs}^{(1)} u_s + \frac{1}{2} \left(\sum_{s=1}^n C_{rs}^{(1)} w_s \right)^2 \right) + A_{11}(x) \left(\sum_{s=1}^n C_{rs}^{(2)} u_s + \sum_{s=1}^n C_{rs}^{(2)} w_s \sum_{s=1}^n C_{rs}^{(1)} w_s \right)$$

$$-l^2 \left[\begin{array}{l} \frac{d^3 A_{11}(x)}{dx^3} \left(\sum_{s=1}^n C_{rs}^{(1)} u_s + \frac{1}{2} \left(\sum_{s=1}^n C_{rs}^{(1)} w_s \right)^2 \right) \\ + 3 \frac{d^2 A_{11}(x)}{dx^2} \left(\sum_{s=1}^n C_{rs}^{(2)} u_s + \sum_{s=1}^n C_{rs}^{(2)} w_s \sum_{s=1}^n C_{rs}^{(1)} w_s \right) \\ + 3 \frac{dA_{11}(x)}{dx} \left(\sum_{s=1}^n C_{rs}^{(3)} u_s + \left(\sum_{s=1}^n C_{rs}^{(2)} w_s \right)^2 + \sum_{s=1}^n C_{rs}^{(3)} w_s \sum_{s=1}^n C_{rs}^{(1)} w_s \right) \\ + A_{11}(x) \left(\sum_{s=1}^n C_{rs}^{(4)} u_s + 3 \sum_{s=1}^n C_{rs}^{(3)} w_s \sum_{s=1}^n C_{rs}^{(2)} w_s + \sum_{s=1}^n C_{rs}^{(4)} w_s \sum_{s=1}^n C_{rs}^{(1)} w_s \right) \end{array} \right]$$

$$= \omega^2 \left[m_0 u_s - (ea)^2 \left(m_0 \sum_{s=1}^n C_{rs}^{(2)} u_s + \frac{\partial^2 m_0}{\partial x^2} u_s + 2 \frac{\partial m_0}{\partial x} \sum_{s=1}^n C_{rs}^{(1)} u_s \right) \right] \quad (29a)$$

$$D_{11} \sum_{s=1}^n C_{rs}^{(4)} w_s + 2 \frac{dD_{11}}{dx} \sum_{s=1}^n C_{rs}^{(3)} w_s + \frac{d^2 D_{11}}{dx^2} \sum_{s=1}^n C_{rs}^{(2)} w_s$$

$$-l^2 \left[D_{11} \sum_{s=1}^n C_{rs}^{(6)} w_s + \frac{d^4 D_{11}}{dx^4} \sum_{s=1}^n C_{rs}^{(2)} w_s + 4 \frac{d^3 D_{11}}{dx^3} \sum_{s=1}^n C_{rs}^{(3)} w_s + 4 \frac{dD_{11}}{dx} \sum_{s=1}^n C_{rs}^{(5)} w_s + 6 \frac{d^2 D_{11}}{dx^2} \sum_{s=1}^n C_{rs}^{(4)} w_s \right] \quad (29b)$$

$$\begin{aligned}
 & -\frac{dA_{11}}{dx} \sum_{s=1}^n C_{rs}^{(1)} u_s - \frac{1}{2} \frac{dA_{11}}{dx} \left(\sum_{s=1}^n C_{rs}^{(1)} w_s \right)^2 - A_{11} \frac{\partial^2 u}{\partial x^2} - A_{11} \sum_{s=1}^n C_{rs}^{(1)} w_s \sum_{s=1}^n C_{rs}^{(2)} w_s \\
 & + (ea)^2 \frac{\partial^2}{\partial x^2} \left[\frac{\partial}{\partial x} \left(A_{11} \sum_{s=1}^n C_{rs}^{(1)} u_s + \frac{1}{2} A_{11} \left(\sum_{s=1}^n C_{rs}^{(1)} w_s \right)^2 \right) \sum_{s=1}^n C_{rs}^{(1)} w_s \right] \\
 & + (l)^2 \frac{\partial}{\partial x} \left[\frac{\partial^2}{\partial x^2} \left(A_{11} \sum_{s=1}^n C_{rs}^{(1)} u_s + \frac{1}{2} A_{11} \left(\sum_{s=1}^n C_{rs}^{(1)} w_s \right)^2 \right) \sum_{s=1}^n C_{rs}^{(1)} w_s \right] \\
 & - (ea)^2 (l)^2 \frac{\partial^2}{\partial x^2} \left[\frac{\partial}{\partial x} \left(\frac{\partial^2}{\partial x^2} \left(A_{11} \sum_{s=1}^n C_{rs}^{(1)} u_s + \frac{1}{2} A_{11} \left(\sum_{s=1}^n C_{rs}^{(1)} w_s \right)^2 \right) \right) \sum_{s=1}^n C_{rs}^{(1)} w_s \right] \\
 & = \omega^2 \left[m_0 w_s - m_1 \sum_{s=1}^n C_{rs}^{(2)} w_s - (ea)^2 \left(\begin{aligned} & m_0 \sum_{s=1}^n C_{rs}^{(2)} w_s + 2 \frac{dm_0}{dx} \sum_{s=1}^n C_{rs}^{(1)} w_s + \frac{d^2 m_0}{dx^2} w_s \\ & - m_1 \sum_{s=1}^n C_{rs}^{(4)} w_s - 2 \frac{dm_1}{dx} \sum_{s=1}^n C_{rs}^{(1)} w_s - \frac{d^2 m_1}{dx^2} \sum_{s=1}^n C_{rs}^{(2)} w_s \end{aligned} \right) \right]
 \end{aligned} \tag{29b}$$

Table 2 Comparison of findings for simply-supported Euler-Bernoulli beam theory with Reddy (2007)

	(ea) ² = 0	(ea) ² = 0.5	(ea) ² = 1	(ea) ² = 2	(ea) ² = 3	(ea) ² = 4
Reddy (2007)	9.8696	9.6347	9.4159	9.0195	8.6693	8.3569
Present study	9.8695892819	9.6347583599	9.4159892578	9.0195697277	8.6693506387	8.35692191826

Table 3 The non-dimensional frequency of a nanotube is compared to that of Shafiei and She (2018) for completely clamped nono-beam theory, (R₀ = L/10 = 2R_i, μ_{e0a} = 4μ₁ = 1)

	Present Euler-Bernoulli theory	Shafiei and She (2018)
First frequency	n = 0	21.62128292
	n = 0.25	17.4608717
	n = 0.5	15.19437084
	n = 0.75	13.83456675
	n = 1	12.94944336
Second frequency	n = 0	43.63616647
	n = 0.25	35.05493773
	n = 0.5	30.43092628
	n = 0.75	28.13400516
	n = 1	25.97828192

Table 4 Comparison of the fundamental normalized frequency in different nonlinear amplitude (Amp), for fully clamped and simply-supported beams

	Simply-Supported		Clamped	
	Present Study	Lestari and Hanagud (2001)	Present Study	Malekzadeh and Shojaee (2013)
Amp = 1	1.089155838	1.0892	1.022200907	1.0222
Amp = 2	1.317761785	1.3178	1.085711201	1.0858
Amp = 3	1.625604381	1.6257	1.183187083	1.1823

Table 5 The nonlinear first frequency ($\sqrt{\psi}$) of axially FG nanotubes against size effect parameters, AFG power indices, and rate of cross-section change under various boundary circumstances are investigated (Amp = 1, SUS304/Si₃N₄)

Boundary condition		Local results	μ _{ea} = 0.0, μ _l = 0.2	μ _{ea} = 0.2, μ _l = 0.0
Simply-Supported	Pure SUS304	β _R = -0.25	2.294709553	2.92939351
		β _R = 0.0	2.156402508	4.04189011
				2.137977284
				2.014125209

Table 5 Continued

Boundary condition			Local results	$\mu_{ea} = 0.0, \mu_l = 0.2$	$\mu_{ea} = 0.2, \mu_l = 0.0$
Simply-Supported	Pure SUS304	$\beta_R = +0.25$	2.567708175	2.436230283	2.3828295
		$\beta_R = -0.25$	2.748089635	3.119976509	2.536773492
Simply Supported-Clamped	Pure SUS304	$\beta_R = 0.0$	2.639723867	2.491311512	2.441268968
		$\beta_R = +0.25$	2.567708175	2.436230283	2.3828295
	AFG, n = 1	$\beta_R = -0.25$	3.212883719	2.699769009	2.948541819
		$\beta_R = 0.0$	3.077768516	2.559195122	2.824406335
		$\beta_R = +0.25$	2.989052174	2.576734858	2.745557945
		$\beta_R = -0.25$	3.383291563	3.104004055	3.101355363
Fully clamped	Pure SUS304	$\beta_R = 0.0$	3.14532407	2.197143468	2.892659873
		$\beta_R = +0.25$	2.989451812	2.705024249	2.761343105
	AFG, n = 1	$\beta_R = -0.25$	3.993972321	1.213023857	3.668727633
		$\beta_R = 0.0$	3.703036773	2.74887766	3.402141914
		$\beta_R = +0.25$	3.511185324	2.577104709	3.228557075

Table 6 The nonlinear second frequency ($\sqrt{\psi}$) of axially FG nanotubes against size effect parameters, AFG power indices, and rate of cross-section change under various boundary circumstances are investigated (Amp = 1, SUS304/Si₃N₄)

Boundary condition			Local results	$\mu_{ea} = 0.0, \mu_l = 0.2$	$\mu_{ea} = 0.2, \mu_l = 0.0$
Simply-Supported	Pure SUS304	$\beta_R = -0.25$	4.581352518	4.72391953	3.769083611
		$\beta_R = 0.0$	4.312635741	0.283124836	3.594786727
		$\beta_R = +0.25$	4.100162794	2.900201601	3.443957362
	AFG, n = 1	$\beta_R = -0.25$	5.453827575	1.727065646	4.459986413
		$\beta_R = 0.0$	5.100944913	1.374862637	4.216866465
		$\beta_R = +0.25$	4.891710744	6.936629554	4.079216256
Simply Supported-Clamped	Pure SUS304	$\beta_R = -0.25$	5.064207498	5.635827744	4.102300233
		$\beta_R = 0.0$	4.782835174	1.101344212	3.904435179
		$\beta_R = +0.25$	4.603414697	1.01272787	3.800179
	AFG, n = 1	$\beta_R = -0.25$	5.990887178	1.307680377	4.815056348
		$\beta_R = 0.0$	5.63225075	6.662609366	4.551141592
		$\beta_R = +0.25$	5.413609436	7.913775222	4.393084865
Fully clamped	Pure SUS304	$\beta_R = -0.25$	5.649978044	6.196687638	4.564692032
		$\beta_R = 0.0$	5.288451346	2.450360558	4.333465519
		$\beta_R = +0.25$	5.017906405	5.529315091	4.159706992
	AFG, n = 1	$\beta_R = -0.25$	6.705821392	9.991160685	5.391498397
		$\beta_R = 0.0$	6.235086123	3.908752762	5.057268585
		$\beta_R = +0.25$	5.934169754	3.408183919	4.860845288

4. Results

Non-dimensional parameters are defined as bellow to understand the results and the results of this study and compare the frequency results to observe the effects of different parameters:

$$\text{Normalized Frequency} = \frac{\text{Nonlinear Frequency}}{\text{Linear Frequency of perfect ceramic tube}} \quad (31a)$$

$$\Psi = \omega \sqrt{\rho_{Ceramic} \times A_{at\ x=0} \times L^4 / E_{Ceramic} \times J_{at\ x=0}} \quad (31b)$$

$$\text{Amp} = a \sqrt{I_{at\ x=0} / A_{at\ x=0}} \quad (31c)$$

$$\mu_l = l/L \quad (31d)$$

$$\mu_{ea} = ea/L \quad (31e)$$

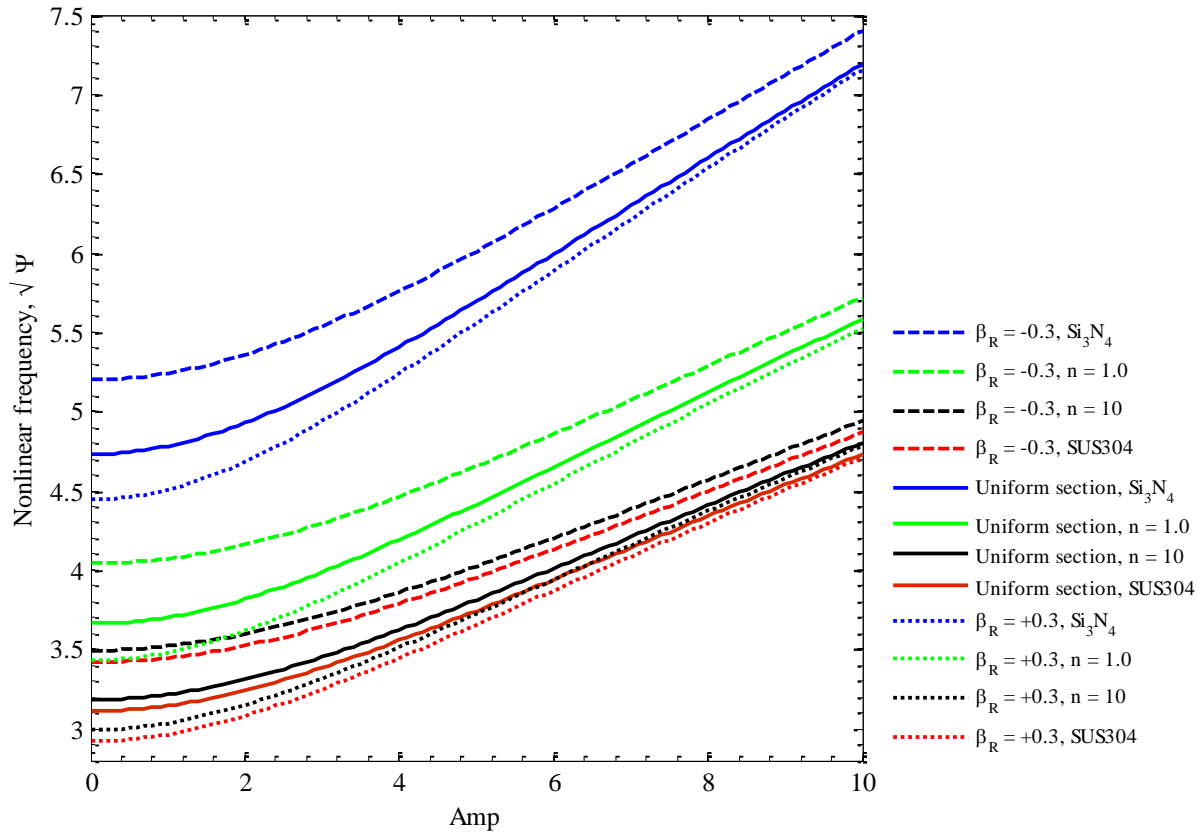


Fig. 2 Fundamental linear (Amp = 0) and nonlinear frequency of clamped nanotube versus Amp, FG power indexes for the various rate of the cross-section, (SUS304/Si₃N₄)

Here, ψ , Amp, μ_l and μ_{ea} are the non-dimensional frequency, nonlinear amplitude, non-dimensional gradient strain parameter and non-dimensional nonlocal parameter. The results are compared to the results of Reddy (2007) where in Table 2 where the convergence of the results show the accuracy of the modeling and solving procedure of this study.

Table 3 shows the validation of the following results through comparing the non-dimensional frequency of fully clamped nano-tube with the results of Shafiei and She (2018). This comparison shows the very good validation between the results, while, they are calculated for two different theories.

The validation of present nonlinear results with the result of Lestari and Hanagud (2001) and Malekzadeh and Shojaee (2013) was shown in Table 4. The normalized frequency of fully clamped and simply supported signals for a variety of nonlinear amplitude (Amp) parameters was shown to be very accurate.

Tables 5 and 6 show the nonlinear first and second frequencies of nickel and SUS304 AFG nano-tubes, respectively. Tables 5 Table 6 investigate many parameters such as the nonlocal and strain gradient parameters, the rate of cross-section change, and AFG power indexes for clamped, simply-supported, and clamped-simply supported. These results indicate that the nonlinear frequency increases as the gradient strain parameter and rate of cross-section change (β_R) increase and decrease, respectively, for all boundary conditions. It is also shown that the nonlinear

frequency lowers when the nonlocal parameter and AFG power indices increase. This is because the effect of the nonlocal and power indices decreases the influence of material stiffness, resulting in a decrease in nonlinear frequency.

Figs. 2 and 3 shows the effect of nonlinear amplitude parameter (Amp) and the parameter of material distribution on the first two frequencies of tubes, for various rates of cross-section change for simply-supported tubes that are made of metal (SUS304) and ceramic (Si₃N₄). It was displayed that the rate of cross-section (β_R) has a significant effect on the linear and nonlinear frequencies, in such a way that, decrease of β_R increases the frequencies due to this fact that the increment of β_R leads to increase of stiffness of the nano-tube which is the cause of frequency decrement. It is clearly shown increases the AFG power indexes because of the higher stiffness of ceramic than metal tends to decreases the frequency. Moreover, the nonlinear frequency increases with Amp, it's because of the increase of solidity of the tube with the Von-Kármán parameter.

Figs. 4 and 5 explain the porosity impact on the first and second frequency of the tube versus the various AFG parameter (n) on the uniform and nonuniform FG tubes. The porosity tends to enhance the frequency due to the hardening phenomenon, on the other hand, the increment of FG indexes decreases the stiffness and leads to a drop in the frequency in both uniform and nonuniform tubes. Moreover, β_R , as the rate of cross-section change, decreases the frequency because of the softening behavior.

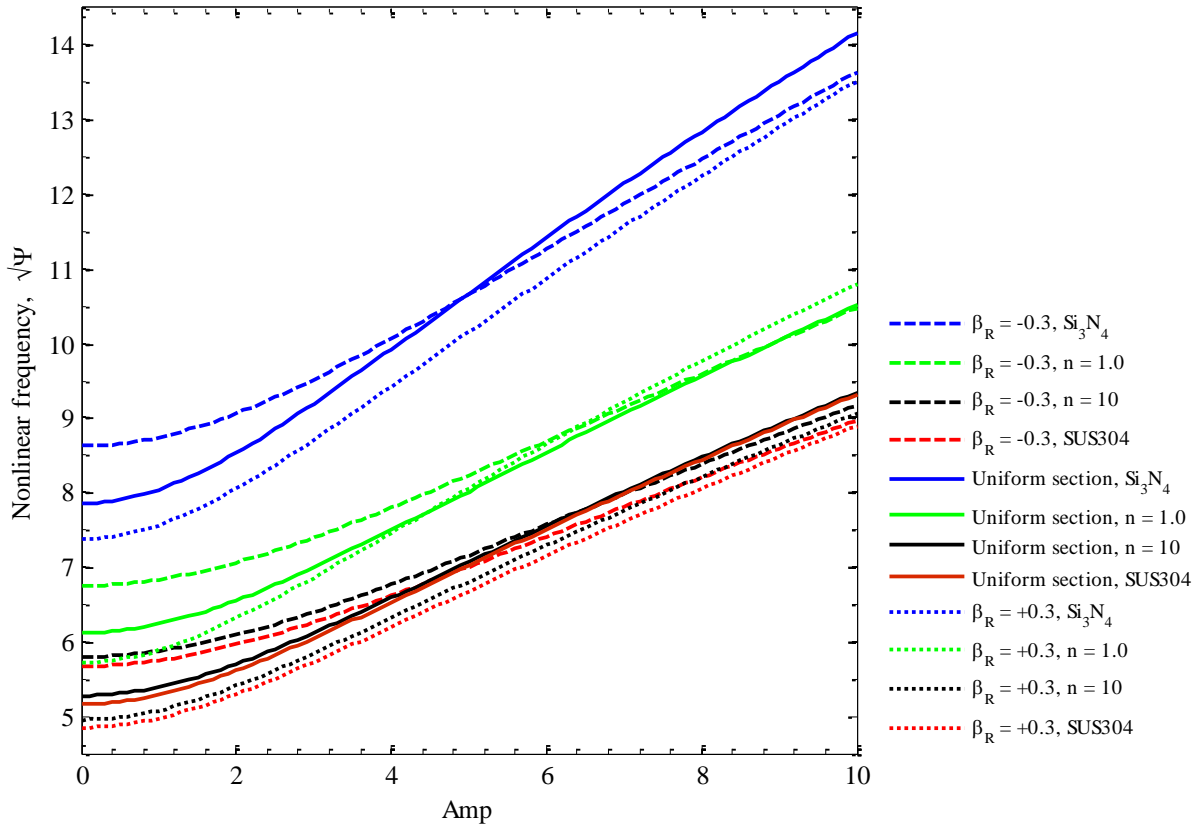


Fig. 3 Second linear (Amp = 0) and nonlinear frequency of clamped nanotube versus Amp, FG power indexes for the various rate of the cross-section, (SUS304/ Si_3N_4)

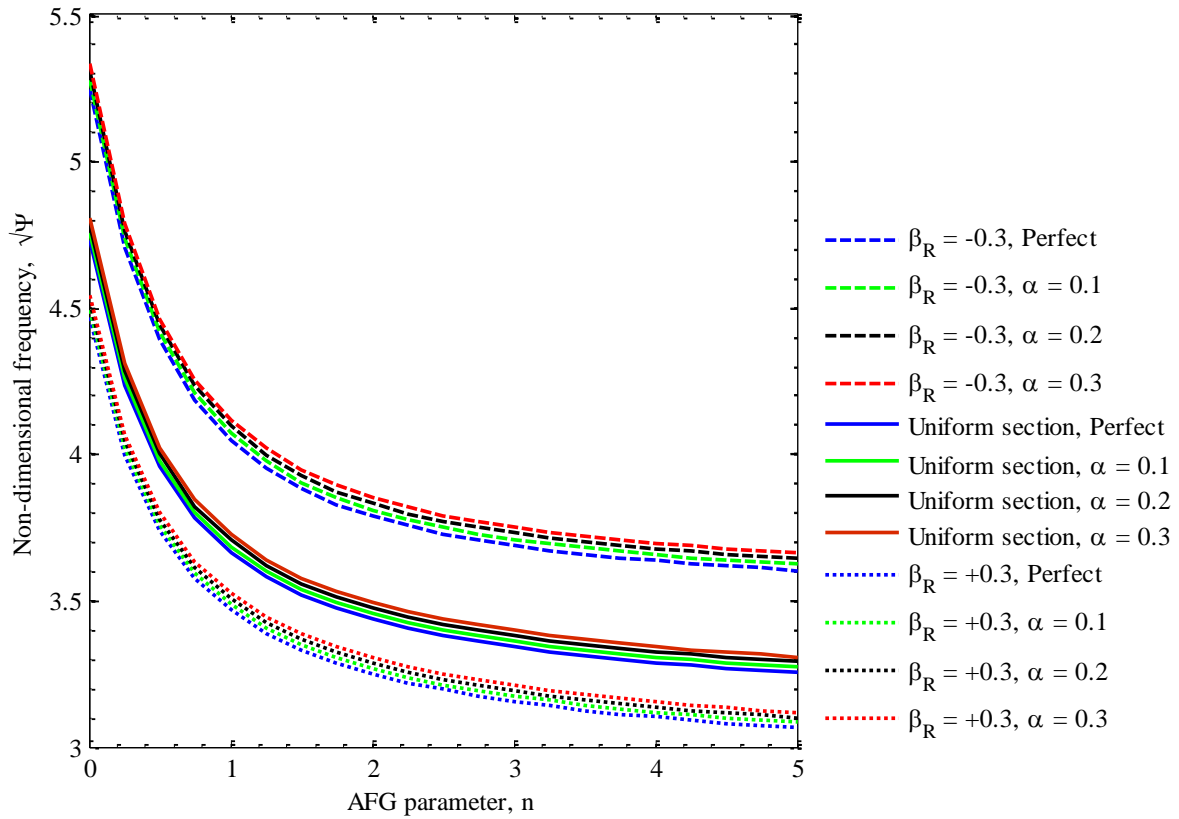


Fig. 4 First frequency of fully clamped nanotube versus AFG power index (n) for the various porosity coefficient (α) and rate of the cross-section (β_R), (SUS304/ Si_3N_4)

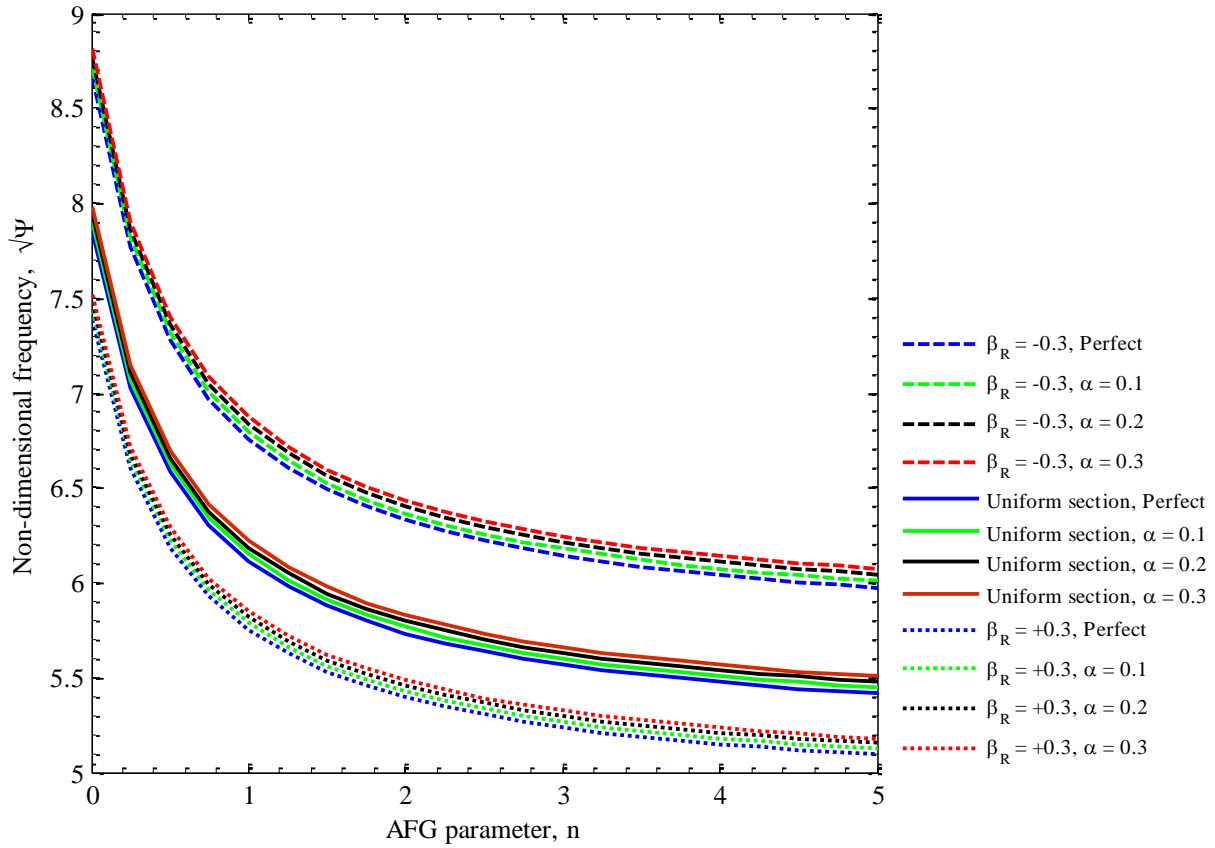


Fig. 5 Second frequency of fully clamped nanotube versus AFG power index (n) for the various porosity coefficient (α) and rate of the cross-section (β_R), (SUS304/Si₃N₄)

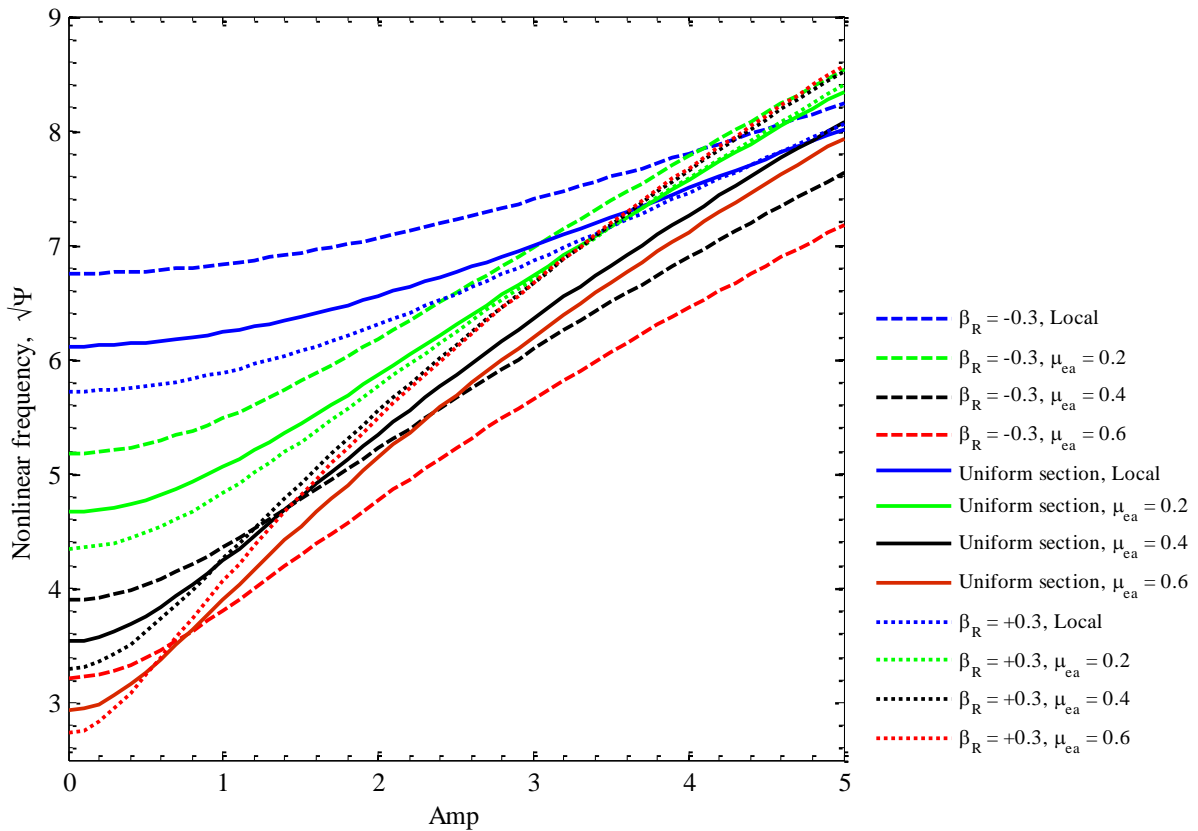


Fig. 6 Fundamental frequency of fully clamped nanotube versus different value of nonlinear amplitude for various parameters of size effect, ($\beta_R = 0.25$, $n = 1$, $\alpha = 0.1$, SUS304/Si₃N₄)

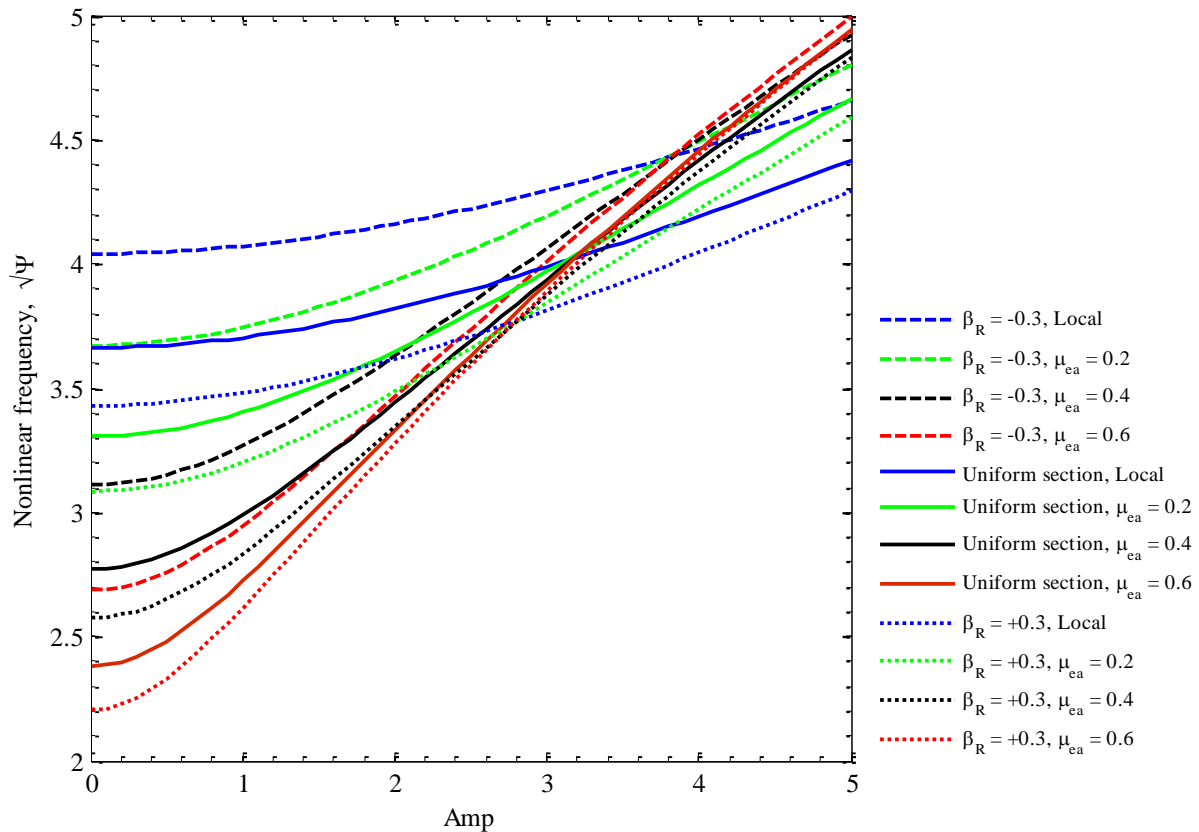


Fig. 7 Second frequency of simply-supported nano-tube versus different value of nonlinear amplitude for various parameters of size effect, ($\beta_R = 0.25, n = 1, \alpha = 0.1, \text{SUS304/Si}_3\text{N}_4$)

Figs. 6 and 7 illustrate the impact of nonlocal factors on the nonlinear first and second frequencies of functionally graded nanotubes as a function of the nonlinear amplitude for both uniform and nonuniform tube sections. Due to the nonlocal parameter's softening effect, both frequencies decrease through the nonlocal parameter, which also lowers the tube stiffness. The nonlinearity of nonlinear amplitude enhances the FG tube's stability and frequency. As previously stated, the cross-section rate (β_R) has a major effect on the dynamic behavior of tubes, and the nonuniformity reduces the stability of FG tubes.

5. Conclusions

In this study, the nonlinear frequency of an axially functionally graded non-uniform nano-tube with porosity is investigated. The Euler-Bernoulli beam model is used to represent the nanotube, and the material composition changes throughout its length. The results in this study are derived utilizing the generalized differential quadrature method, and the nonlocal strain gradient, as well as Eringen's nonlocal theory, are considered. The findings are examined for various nonlinearities, nonlocal and strain gradient parameters, AFG power indices, porosity parameter, and different non-uniformities of the nano-tube for clamped, simply supported, and clamped-simply supported boundary conditions. The main results covered in this research are as follows:

- The frequency increases with nonlinearity for all of the boundary conditions stated.
- As the stiffness increases with the gradient strain parameter, the frequency increases as well.
- The nonlocal parameter, unlike the gradient strain parameter, decreases stiffness and hence frequency.
- The nonlocal and strain gradient parameters have no impact on each other.
- When these parameters are raised, the effect of both nonlocal and strain gradient components increases. To put it another way, the effects of these variables are not linear.
- The frequency decreases as the stiffness decreases, as do the AFG power indices.
- With this kind of porosity, the frequency increases.
- As β_R rises, the frequency decreases.
- For clamped, simply supported, and clamped-simply supported boundary conditions, the studied parameters have the same effect.

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