

Non-local orthotropic elastic shell model for vibration analysis of protein microtubules

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Abstract. Vibrational analysis in microtubules is examined based on the nonlocal theory of elasticity. The complete analytical formulas for wave velocity are obtained and the results reveal that the small scale effects can reduce the frequency, especially for large longitudinal wave-vector and large circumferential wave number. It is seen that the small scale effects are more significant for smaller wave length. The methods and results may also support the design and application of nano devices such as micro sound generator etc. The effects of small scale parameters can increase vibrational frequencies of the protein microtubules and cannot be overlooked in the analysis of vibrating phenomena. The results for different modes with nonlocal effect are checked.

Keywords: microtubules; nonlocal theory of elasticity; vibration analysis; small scale effects

1. Introduction

Animals have skeletal system for the support and maintenance of their bodies. Cell is the functional and basic structural unit of the life and therefore this unit should have some skeletal structure for its rigidity and support. All types of cells have skeleton called cytoskeleton. One of the components of cytoskeleton is microtubules (MTs). MTs are like hollow cylinder having outer and inner diameter 30 nanometer (nm) and 20 nanometer (nm) respectively with length up to 100 μm or more (de Pablo *et al.* 2003). MTs are organized by 13 parallel protofilaments (PFs), linked in circular shape (Nogales 2001). These filaments are composed of $\alpha\beta$ -tubulin dimers (Gittes *et al.* 1993).

The configuration of MTs can also be illuminated by a pair of integers, N and S. N is PFs number and S is helix start number. The best common type of MTs is assembled in vivo where the PFs run parallel to longitudinal axis (Ishida *et al.* 2007). Sofiyev *et al.* (2006) considered the dynamic stability problem of a cylindrical shell composed of non-homogeneous orthotropic materials with Young's moduli and density varying continuously in the thickness direction under the effect of an axial compressive load varying with a parabolic function of time. At first, the fundamental relations and the modified Donnell type dynamic stability equations of a non-homogeneous orthotropic cylindrical

shell are set up. Batou *et al.* (2019) studied the wave propagations in sigmoid functionally graded (S-FG) plates using new Higher Shear Deformation Theory (HSDT) based on two-dimensional (2D) elasticity theory. The current higher order theory has only four unknowns, which mean that few numbers of unknowns, compared with first shear deformations and others higher shear deformations theories and without needing shear corrector. MTs are the best rigid filaments of all types of cytoskeleton which are mostly responsible for shape and mechanical inelasticity of the cell and basically involved in large range of functions such as cell division, intercellular transportation and the cell motility (Howard 2001, Boal 2012). Tohidi *et al.* (2017) investigated the dynamic stability of embedded functionally graded (FG)-carbon nanotubes (CNTs)-reinforced micro cylindrical shells. The structure is subjected to harmonic non-uniform temperature distribution and 2D magnetic field. The CNT reinforcement is either uniformly distributed or FG along the thickness direction where the effective properties of nano-composite structure are estimated through Mixture law. Their mechanical characteristics show an essential character in various cellular functions. MTs are tracks for the motor protein together with they change (Fernández and Vico 2011). They help in the movement of chromosomes, during the cell division also form moving cores of flagella and cilia and (Shaw *et al.* 2000). Along PFs, longitudinal bonds are much stronger than lateral bonds between adjacent PFs of MTs (Needleman *et al.* 2005). The elastic moduli along longitudinal and circumferential direction differ remarkably. It is greater in former case and lesser in later one (Jánosi *et*

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al. 2002). Narwariya *et al.* (2018) presented the vibration and harmonic analysis of orthotropic laminated composite plate. The response of plate is determined using Finite Element Method. The eight noded shell 281 elements are used to analyze the orthotropic plates and results are obtained so that the right choice can be made in applications such as aircrafts, rockets, missiles, etc. to reduce the vibration amplitudes. Hussain *et al.* (2017) demonstrated an overview of Donnell theory for the frequency characteristics of two types of SWCNTs. Fundamental frequencies with different parameters have been investigated with wave propagation approach. Arefi (2018) studied the nonlocal free vibration analysis of a doubly curved piezoelectric nano shell. First order shear deformation theory and nonlocal elasticity theory is employed to derive governing equations of motion based on Hamilton's principle. The doubly curved piezoelectric nano shell is resting on Pasternak's foundation. Hussain and Naeem (2017) examined the frequencies of armchair tubes using Flügge's shell model. The effect of length and thickness-to-radius ratios against fundamental natural frequency with different indices of armchair tube was investigated. Salah *et al.* (2019) examined a simple four-variable integral plate theory is employed for examining the thermal buckling properties of functionally graded material (FGM) sandwich plates. The proposed kinematics considers integral terms which include the effect of transverse shear deformations. Hussain and Naeem (2018a) used Donnell's shell model to calculate the dimensionless frequencies for two types of single-walled carbon nanotubes. The frequency influence was observed with different parameters. Along longitudinal direction, elastic modulus is also greater in magnitude than the shear modulus (de Pablo *et al.* 2003). No doubt, microfilaments of MTs form a helical structure but this helix is not responsible for the mechanical behavior of MTs (Kasas *et al.* 2004). Fatahi-Vajari *et al.* (2019) studied the vibration of single-walled carbon nanotubes based on Galerkin's and homotopy method. This work analyses the nonlinear coupled axial-torsional vibration of single-walled carbon nanotubes (SWCNTs) based on numerical methods. Two-second order partial differential equations that govern the nonlinear coupled axial-torsional vibration for such nanotube are derived. Asghar *et al.* (2019a, b) conducted the vibration of nonlocal effect for double-walled carbon nanotubes using wave propagation approach. Many material parameters are varied for the exact frequencies of many indices of double-walled carbon nanotubes. Shahrma *et al.* (2019) studied the functionally graded material using sigmoid law distribution under hygrothermal effect. The Eigen frequencies are investigated in detail. Frequency spectra for aspect ratios have been depicted according to various edge conditions. Benmansour *et al.* (2019) analyzed the dynamic and bending behaviors of isolated protein microtubules. Microtubules (MTs) can be considered as bio-composite structures that are elements of the cytoskeleton in eukaryotic cells and possess considerable roles in cellular activities. They have higher mechanical characteristics such as superior flexibility and stiffness. Particularly, along longitudinal direction, elastic modulus is higher than the shear modulus of the MTs and also elastic

modulus along the longitudinal direction is higher than the elastic modulus along with the circumferential directions (Kasas *et al.* 2004, Tuszyński *et al.* 2005). These outcomes provide the basis for the development of a new model named "Orthotropic Elastic Shell Model" for the observation of different mechanical behaviour of MTs (Wang *et al.* 2006a, b). Rezaiee-Pajand *et al.* (2018) investigated the static behavior of non-prismatic sandwich beams composed of functionally graded (FG) materials for the first time. Two types of beams in which the variation of elastic modulus follows a power-law form are studied. The principle of minimum total potential energy is applied along with the Ritz method to derive and solve the governing equations. Sofiyev *et al.* (2009) truncated the free vibration of non-homogeneous truncated conical shells on a Winkler foundation. After formed the fundamental relations and governing equations, the dimensionless frequency parameter of the non-homogeneous isotropic truncated conical shell with or without an elastic foundation are found. Inspired by these ideas, an "Orthotropic shell model" was established to study the propagation of the wave along MTs (Qian *et al.* 2007). A respectable agreement has been made between this model and the available discrete model and the experiments. Ayat *et al.* (2018) resulted the use of optimum content of supplementary cementing materials (SCMs) such as limestone filler (LF) to blend with Portland cement in many environmental and technical advantages, such as increase in physical properties, enhancement of sustainability in concrete industry and reducing CO2 emission are well known. Non-local theory of elasticity is pioneered by Eringen, states that the stress at a point is not simply the function of strain on that point but it also depends on long range interactions of the surrounding areas. Using nonlocal theory of elasticity, some mathematicians (Gao and An 2010, An and Gao 2010, Gao and Lei 2009) showed that the persistence length of MTs, which is the direct demonstration of the microtubules' stiffness, is reformed by considering the small scale effects into the account. The effects of small scale are also affect the buckling, post-buckling and bending of MTs (Civalek and Demir 2011, Civalek and Demir 2016, Shen 2010). Behera and Kumari (2018) developed an exact solution for free vibration of the Levy-type rectangular laminated plate considering the most efficient Zig-Zag theory (ZIGT) and third order theory (TOT). The plate is subjected to hard simply supported boundary condition (Levy-type) along x axis. Using the equilibrium equations and the plate constitutive relations, a set of 12 m first order differential homogenous equations are obtained. Civalek *et al.* (2010) formulated the equations of motion and bending of Euler-Bernoulli beam using the nonlocal elasticity theory for cantilever microtubules (MTs). The method of differential quadrature (DQ) has been used for numerical modeling. The size effect is taken into consideration using the Eringen's non-local elasticity theory. Jamali *et al.* (2019) devoted to study the post-buckling analysis of functionally graded carbon nanotubes reinforced composite (FG-CNTRC) micro plate with cut out subjected to magnetic field and resting on elastic medium. The basic formulation of plate is based on first order shear deformation theory

(FSDT) and the material properties of FG-CNTRCs are presumed. Many mathematicians have been discoursed the vibrations of MTs by considering small scale effects and they showed that these effects have a notable impact on the dynamical properties of the MTs (Heireche *et al.* 2010), (Ansari and Arash, 2013). Recently Hussain and Naeem (2019a, b, c, d, 2020a) and performed the vibration of SWCNTs based on wave propagation approach and Galerkin's method. They investigated many physical parameters for the rotating and non-rotating vibrations of armchair, zigzag and chiral indices. Moreover, the mass density effect of single walled carbon nanotubes with in-plane rigidity has been calculated for zigzag and chiral indices.

Earlier work in literature was limited for the circumferential vibrations of MTs (Wang and Zhang 2008), using orthotropic shell model. In that study, non-local shear deformable Shell model is used to study non-linear vibration analysis of MTs. The numerical results of this work showed that fundamental frequency of MTs are very complex to small scale parameter. This result reveals that small scale parameter decrease the natural frequencies of the MTs but the effect of small scale parameter on non-linear vibration reply is relatively feeblor when compared to small scale parameter effects on post reply of the MTs.

2. Materials and methods

Over the past several years vibration of nano and micro structures of various configurations and boundary conditions have been extensively studied (Hussain *et al.* 2018a, Hussain *et al.* 2018b, Hussain *et al.* 2018c, Hussain and Naeem 2018b, Hussain *et al.* 2019a, Hussain and Naeem 2020b, Hussain *et al.* 2019b, Hussain *et al.* 2020a, Sehar *et al.* 2020, Hussain *et al.* 2020b, c, d). Here nonlocal rthotropic Elastic Shell Model is utilized to analyze the vibration of MTs. Nonlocal orthotropic Elastic shell model is developed here to investigate the vibration of protein MTs and the wave propagation approach is applied to find the dispersion relations for free MTs.

2.1 Nonlocal orthotropic shell model for free MTs

In the “non-local theory” of the elasticity, the effects of the small scale parameters are not negligible in the orthotropic modeling of shells, which are elastic in nature. The constitutive relationships of three-dimensional problems take the form given below (Eringen 2002).

$$(1 - (e_o a)^2 \nabla^2) \sigma_{ij} = C_{ijkl} \varepsilon_{kl} \quad (1)$$

Where a is internal length (similar to length of Carbon-Carbon bond etc.), and e_o designated as a material constant depending on the nature of materials; and hence $e_o a$ is the small scale parameter ε_{ij} and σ_{ij} are the strain and stress tensors correspondingly. The C_{ijkl} , represents elastic modulus tensor of local elasticity. The organelle MTs may be taking in consideration as a tube which is circular cylindrical in shape. The depth of bridge is round about 1.1 nm, equivalent depth h is round about 2.7 nm. The

effective depth h_o for bending is round about 1.6 nm (Qian *et al.* 2007, Gao and An 2010).

The following relations given by Flugge's shell theory (Wang and Carter 2002, Zou and Foster 1995)

$$\varepsilon_\alpha = \frac{1}{R} \left(\frac{\partial u}{\partial \alpha} - \gamma \frac{\partial^2 w}{\partial^2 \alpha} \right). \quad (2)$$

$$\varepsilon_\beta = \frac{1}{R} \left(\frac{\partial v}{\partial \beta} + w \right) - \frac{\gamma}{R(1+\gamma)} \left(\frac{\partial^2 w}{\partial \beta^2} + w \right) \quad (3)$$

$$\varepsilon_{\alpha\beta} = \frac{\gamma}{R(1+\gamma)} \left[\frac{\partial u}{\partial \beta} + \frac{\partial v}{\partial \alpha} + 2\gamma \left(\frac{\partial v}{\partial \alpha} - \frac{\partial^2 w}{\partial \alpha \partial \beta} \right) + \gamma^2 \left(\frac{\partial v}{\partial \alpha} - \frac{\partial^2 w}{\partial \alpha \partial \beta} \right) \right] \quad (4)$$

The strain-stress relationship by Hook's law in dimensionless coordinates derived from Eq. (1) is as under (Gao and An 2010).

$$\sigma_\alpha - (e_o a)^2 \nabla^2 \sigma_\alpha = E_1 \frac{(\varepsilon_\alpha + \mu_1 \varepsilon_\beta)}{(1 - \mu_1 \mu_2)}, \quad (5)$$

$$\sigma_\beta - (e_o a)^2 \nabla^2 \sigma_\beta = E_2 \frac{(\mu_2 \varepsilon_\alpha + \varepsilon_\beta)}{(1 - \mu_1 \mu_2)}, \quad (6)$$

$$\tau_{\alpha\beta} - (e_o a)^2 \nabla^2 \tau_{\alpha\beta} = G \varepsilon_{\alpha\beta}. \quad (7)$$

where σ_α , σ_β and $\tau_{\alpha\beta}$ are normal and shear stresses, and ε_α , ε_β and $\varepsilon_{\alpha\beta}$ are respective strains; E_1 and E_2 are moduli of elasticity; μ_2 and μ_1 are Poisson's ratios in the directions of α and β correspondingly. G is modulus of rigidity or shear modulus. Also we have $E_1 \mu_1 = E_2 \mu_2$ and $\nabla^2 = (\partial^2 / \partial \alpha^2 + \partial^2 / \partial \beta^2) / R^2$ which is the Laplace operator in dimensionless coordinates.

The elements of shell in our coordinates are, N, S, Q which are the stress resultants and M is the moment. The thermal expansion causes pre-stress, which is neglected due to the presence of temperature and considered by means of reference temperature. We arrive at dynamical equilibrium equations.

$$\frac{\partial N_\alpha}{\partial \alpha} + \frac{\partial S_\beta}{\partial \beta} + k = \rho h R \frac{\partial^2 u}{\partial t^2} \quad (8)$$

$$\frac{\partial N_\beta}{\partial \beta} + \frac{\partial S_\alpha}{\partial \alpha} + Q_\beta = \rho h R \frac{\partial^2 v}{\partial t^2}, \quad (9)$$

$$\frac{\partial Q_\sigma}{\partial \alpha} + \frac{\partial Q_\beta}{\partial \beta} - N_\beta = \rho h R \frac{\partial^2 w}{\partial t^2}, \quad (10)$$

$$\frac{\partial M_{\alpha\beta}}{\partial \alpha} + \frac{\partial M_\beta}{\partial \beta} - R Q_\beta = 0, \quad (11)$$

$$\frac{\partial M_{\beta\alpha}}{\partial \beta} + \frac{\partial M_\alpha}{\partial \alpha} - R Q_\alpha = 0. \quad (12)$$

Where ρ is mass density.

The including resultants (N, S, Q) which are derived from overhead set of equations by using stress components.

$$(1 - (e_o a)^2 \nabla^2) \begin{bmatrix} N_\alpha & S_\alpha \\ M_\alpha & M_{\alpha\beta} \end{bmatrix} = \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{bmatrix} \sigma_\alpha & \tau_{\alpha\beta} \\ z \sigma_\alpha & z \tau_{\alpha\beta} \end{bmatrix} \left(1 + \frac{z}{R} \right) dz, \quad (13)$$

$$(1 - (e_o a)^2 \nabla^2) \begin{bmatrix} N_\beta & S_\beta \\ M_\beta & M_{\beta\alpha} \end{bmatrix} = \int_{-\frac{h}{2}}^{\frac{h}{2}} \begin{bmatrix} \sigma_\beta & \tau_{\beta\alpha} \\ z \sigma_\beta & z \tau_{\beta\alpha} \end{bmatrix} dz, \quad (14)$$

$$(1 - (e_o a)^2 \nabla^2) (Q_\alpha, Q_\beta) = \int_{-\frac{h}{2}}^{\frac{h}{2}} [\tau_{\alpha z}, \tau_{\beta z}] dz. \quad (15)$$

Where h is thickness of the shell. Above equations result in

$$N_\alpha - (e_0 a)^2 \nabla^2 N_\alpha = \frac{K}{R} \left[\frac{\partial u}{\partial \alpha} + \mu_1 \left(\frac{\partial v}{\partial \beta} + w \right) - c^2 \frac{\partial^2 w}{\partial \alpha^2} \right], \quad (16)$$

$$N_\beta - (e_0 a)^2 \nabla^2 N_\beta = \frac{K k_1}{R} \left[\frac{\partial v}{\partial \beta} + \mu_2 \frac{\partial u}{\partial \alpha} + w + c^2 \left(\frac{\partial^2 w}{\partial \beta^2} + w \right) \right] \quad (17)$$

$$S_\alpha - (e_0 a)^2 \nabla^2 S_\alpha = \frac{K k_2}{R} \left[\frac{\partial u}{\partial \beta} + \frac{\partial v}{\partial \alpha} - c^2 \left(\frac{\partial^2 w}{\partial \alpha \partial \beta} - \frac{\partial v}{\partial \alpha} \right) \right], \quad (18)$$

$$S_\beta - (e_0 a)^2 \nabla^2 S_\beta = \frac{K k_2}{R} \left[\frac{\partial u}{\partial \beta} + \frac{\partial v}{\partial \alpha} + c^2 \left(\frac{\partial^2 w}{\partial \alpha \partial \beta} + \frac{\partial v}{\partial \alpha} \right) \right], \quad (19)$$

$$M_\alpha - (e_0 a)^2 \nabla^2 M_\alpha = -K c^2 \left[\frac{\partial u}{\partial \alpha} + \mu_1 \frac{\partial v}{\partial \beta} - \left(\frac{\partial^2 w}{\partial \alpha^2} + \mu_1 \frac{\partial w}{\partial \beta^2} \right) \right], \quad (20)$$

$$M_\beta - (e_0 a)^2 \nabla^2 M_\beta = K k_1 c^2 \left[\frac{\partial^2 w}{\partial \beta^2} + w + \mu_2 \frac{\partial^2 w}{\partial \alpha^2} \right], \quad (21)$$

$$M_{\alpha\beta} - (e_0 a)^2 \nabla^2 M_{\alpha\beta} = 2K k_2 c^2 \left[\frac{\partial v}{\partial \alpha} - \frac{\partial^2 w}{\partial \alpha \partial \beta} \right], \quad (22)$$

$$M_{\beta\alpha} - (e_0 a)^2 \nabla^2 M_{\beta\alpha} = K k_2 c^2 \left[\frac{\partial u}{\partial \beta} - \frac{\partial v}{\partial \alpha} + 2 \frac{\partial^2 w}{\partial \alpha \partial \beta} \right], \quad (23)$$

$$\begin{aligned} Q_\alpha - (e_0 a)^2 \nabla^2 Q_\alpha = \\ \frac{K c^2}{R} \left[\frac{\partial^2 u}{\partial \alpha^2} - k_2 \frac{\partial^2 v}{\partial \beta^2} + (k_2 + \mu_1) \frac{\partial^2 v}{\partial \alpha \partial \beta} - \frac{\partial^3 w}{\partial \alpha^3} - \right. \\ \left. (2k_2 + \mu_1) \frac{w}{\partial \alpha \partial \beta^2} \right], \quad (24) \end{aligned}$$

$$\begin{aligned} Q_\beta - (e_0 a)^2 \nabla^2 Q_\beta = \\ \frac{K k_1 c^2}{R} \left[2 \frac{k_2}{k_1} \frac{\partial^2 v}{\partial \alpha^2} - \frac{\partial^3 w}{\partial \beta^3} - \frac{\partial w}{\partial \beta} - (2 \frac{\mu_1}{k_1} + \mu_2) \frac{\partial^3 w}{\partial \alpha^2 \partial \beta} \right]. \quad (25) \end{aligned}$$

Where $K = \frac{E_1 h}{(1 - \mu_1 \mu_2)}$, $k_1 = \frac{E_2}{E_1}$, $k_2 = G(1 - \mu_1 \mu_2)/E_1$, $c^2 = \frac{h_0^2}{R^2 h}$

$$\begin{aligned} & \left[\frac{\partial^2}{\partial \alpha^2} + k_2(1 + c^2) \frac{\partial^2}{\partial \beta^2} \right] u + \left[(\mu_1 + k_2) \frac{\partial^2}{\partial \alpha \partial \beta} \right] v \\ & + \left[\mu_1 \frac{\partial}{\partial \alpha} + c^2 \left(k_2 \frac{\partial^3}{\partial \alpha \partial \beta^2} - \frac{\partial^3}{\partial \alpha^3} \right) \right] w = \frac{\rho h R^2 (1 - (e_0 a)^2 \nabla^2)}{K} \frac{\partial^2 u}{\partial t^2} \quad (26) \end{aligned}$$

$$\begin{aligned} & \left[(\mu_1 + k_2) \frac{\partial^2}{\partial \alpha \partial \beta} \right] u + \left[k_2(1 + 3c^2) \frac{\partial^2}{\partial \alpha^2} + k_1 \frac{\partial^2}{\partial \beta^2} \right] v + \\ & \left[k_1 \frac{\partial}{\partial \beta} - c^2 (\mu_1 + 3k_2) \frac{\partial^3}{\partial \alpha^2 \partial \beta} \right] w = \frac{\rho h R^2 (1 - (e_0 a)^2 \nabla^2)}{K} \frac{\partial^2 v}{\partial t^2}. \quad (27) \end{aligned}$$

$$\begin{aligned} & \left[\mu_1 \frac{\partial}{\partial \alpha} - c^2 \left(\frac{\partial^3}{\partial \alpha^3} - k_2 \frac{\partial^3}{\partial \alpha \partial \beta^2} \right) \right] u + \left[k_1 \frac{\partial}{\partial \beta} - c^2 (\mu_1 + 3k_2) \frac{\partial^3}{\partial \alpha^2 \partial \beta} \right] v + \\ & \left[\left(1 + \frac{1}{c^2} \right) k_1 + \frac{\partial^4}{\partial \alpha^4} + k_1 \frac{\partial^4}{\partial \beta^4} + 2k_1 \frac{\partial^2}{\partial \beta^2} + (2\mu_1 + 4k_2) \frac{\partial^4}{\partial \alpha^2 \partial \beta^2} \right] w = - \frac{\rho h R^2 (1 - (e_0 a)^2 \nabla^2)}{K} \frac{\partial^2 w}{\partial t^2} \quad (28) \end{aligned}$$

2.2 Boundary conditions

Three kinds of boundary condition may be assumed while solving such a problems (Ansari and Arash, 2013). These three conditions are:

✓ Clamped-Clamped

$$\alpha = \beta = \gamma = \frac{\partial \gamma}{\partial \alpha} = 0 \quad \text{at} \quad \alpha = 0, \alpha = \frac{L}{R} \quad (29)$$

✓ Clamped-free

$$\alpha = \beta = \gamma = \frac{\partial \gamma}{\partial \alpha} = 0 \quad \text{at} \quad \alpha = 0,$$

✓ Simply supported- Simply supported

$$\beta = \gamma = N_{\alpha\alpha} = 0, \quad \text{at} \quad \alpha = 0, \alpha = \frac{L}{R}. \quad (30)$$

$$N_{\alpha\alpha} = M_{\alpha\alpha} = N_{\alpha\beta} = M_{\alpha\beta} = 0 \quad \text{at} \quad \alpha = \frac{L}{R}. \quad (31)$$

L is the length of microtubules.

3. General solutions of vibrations

In case of non- axisymmetric vibration, the solutions of Eqs (26-28) with boundary conditions (29-31) are given by

$$\begin{aligned} u(\alpha, \beta, t) &= U \sin k\alpha. \cos n\beta e^{i\omega t}, \\ v(\alpha, \beta, t) &= V \cos k\alpha. \sin n\beta e^{i\omega t}, \\ w(\alpha, \beta, t) &= W \cos k\alpha. \cos n\beta e^{i\omega t}. \quad (32) \end{aligned}$$

Where U , V and W are waves amplitudes of the vibration with the direction of x , y and z correspondingly. Wave vecto $k = \frac{\pi m R}{L}$ is dimensionless in the longitudinal direction m is the half wave number in the longitudinal direction and ω is the frequency of vibration Putting Eq. (32) in Eqs. (26)-(28) will give us the following three homogeneous equations.

$$\begin{aligned} & \left[(1 + c^2) k_2 n^2 + k^2 - \frac{\eta h \rho \omega^2 R^2}{K} \right] U + n k (k_2 + \mu_1) \\ & V + K [c^2 (k^2 - k_2 n^2) + \mu_1] W = 0, \\ & n k \left(\frac{k_2}{\mu_1} \right) U + \left[(1 + 3c^2) k_2 k^2 + k_1 n^2 - \frac{\eta h \rho \omega^2 R^2}{K} \right] V \\ & + n [k_1 + c^2 k^2 (3k_2 + \mu_1)] W = 0, \\ & k [c^2 (k^2 - k_2 n^2) + \mu_1] U + n [k_1 + c^2 k^2 (3k_2 + \mu_1)] \\ & V + [(1 + c^2) k_1 + c^2 [k^4 + k_1 n^2 (n^2 - 2) + 2k^2 n^2 \\ & (3k_2 + \mu_1)] - \frac{\eta h \rho \omega^2 R^2}{K} \right] W = 0. \quad (33) \end{aligned}$$

Where $\eta = 1 + (n^2 + k^2) \left(\frac{e_0 a}{R} \right)^2$

Writing the above set of equation (33) in the matrix form as

$$[H^{(1)}(k, \omega)]_{3 \times 3} \begin{bmatrix} U \\ V \\ W \end{bmatrix} = [0 \quad 0 \quad 0]^T, \quad (34)$$

Nonzero solution of above system exist if

$$\text{Det}[H^{(1)}(k, \omega)]_{3 \times 3} = 0. \quad (35)$$

Eq. (35) is written as

$$\Omega^3 + \Omega^2 e + \Omega f + g = 0 \quad (36)$$

Where

$$\Omega = \frac{\eta h \rho \omega^2 R^2}{K}, \quad e = -(a_{11} + a_{22} + a_{33}),$$

$$f = (a_{11} a_{22} + a_{11} a_{33} + a_{22} a_{33}) - (a_{12}^2 + a_{23}^2 + a_{13}^2),$$

$$g = (a_{11} a_{23}^2 + a_{22} a_{13}^2 + a_{33} a_{23}^2) + 2a_{11} a_{12} a_{13} - a_{11} a_{22} a_{33}.$$

And in the above equations

$$a_{11} = k^2 + (1 + c^2) k_2 n^2, \quad a_{12} = -k n (k_2 + \mu_1),$$

$$a_{13} = k [(k^2 + k_2 n^2) c^2 + \mu_1]$$

$$a_{21} = 0, \quad a_{22} = k^2 (1 + 3c^2) + n^2 k_1,$$

$$a_{23} = n k_1 + n c^2 k^2 (\mu_1 + 3k_2),$$

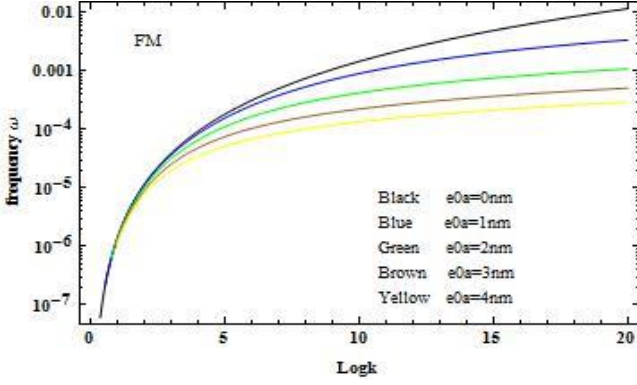


Fig. 1 Effect of non-local parameter (e_0a) on the vibrational frequency for non-axisymmetric vibrations ($n=1$) for first mode

$$a_{31} = 0, \quad a_{32} = 0, \quad a_{33} = (1 + c^2)k_1 + c^2[k^4 + n^2k_1(n^2 - 2) + 2k^2n^2(2k_2 + \mu_1)].$$

From Eq. (36), after long calculations and heavy simplifications, we get the analytic formulas for wave velocity

$$\omega_1^2 = \left(\sqrt[3]{T + \sqrt{D}} + \sqrt[3]{T - \sqrt{D}} - e/3 \right) K / (h\eta\rho R^2), \quad (37)$$

$$\omega_2^2 = \left(\lambda_1 \sqrt[3]{T + \sqrt{D}} + \lambda_2 \sqrt[3]{T - \sqrt{D}} - e/3 \right) K / (h\eta\rho R^2), \quad (38)$$

$$\omega_3^2 = \left(\lambda_2 \sqrt[3]{T + \sqrt{D}} + \lambda_1 \sqrt[3]{T - \sqrt{D}} - e/3 \right) K / (h\eta\rho R^2). \quad (39)$$

Where $D = T^2 + \left(\frac{f}{3} - \frac{e^2}{9}\right)^3$, $T = -\frac{e^3}{27} + \frac{ef}{6} - \frac{g}{2}$, $\lambda_1 = -\frac{1+\sqrt{3}}{2}$, $\lambda_2 = -\frac{1-i\sqrt{3}}{2}$ and $i = \sqrt{-1}$ is imaginary unit.

In the expression of wave speed the small scale parameter e_0a occurs only in η so η can visualize the small scale effects that are affected by long range relation. $\eta = 1$ Refers to the classical case because $\eta = 1$ for $e_0a = 0$. If we take $e_0a = 0$, the equations will degrade to the local theory (Qian *et al.* 2007). For nonlocal cases η takes the values greater than 0.

4. Results and discussion

In this section we will explain the vibrational frequency designed against wave vector graphically. We will elaborate the circumferential half wave number and small scale effect on the wave speed. In order to analyze the effects of small scale, present study is independent of time. The material parameters of microtubules can be obtained by theoretical and experimental methods and few of them are précised (Wang and Gao 2016). In our discussions, we have $E_1 = 1 \times 10^9$ Pa, $E_2 = 1 \times 10^6$ Pa, $\mu_2 = 0.3$, $G = 1 \times 10^6$ Pa. e_0a be influenced by many features. In order to check the effects of the e_0a (Small scale parameter), we have taken the range of e_0a between 0 to 4 nm. Fig. 1, shows that for the first mode of frequency, there is a slight difference between classical and nonlocal results, if we give the values to the wave vector (k) from 0 to 3.5. But there exist a significant difference between the both classical and non-local results, if we give the values to the wave vector

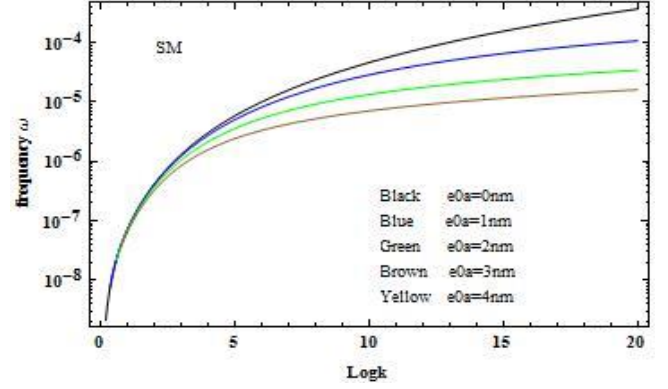


Fig. 2 Effect on non-local parameter (e_0a) on the vibrational frequency for non-axisymmetric vibrations ($n=1$) for second mode of frequency

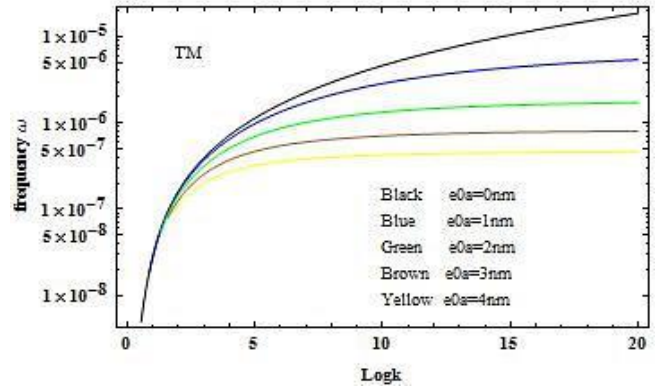


Fig. 3 Effect on non-local parameter (e_0a) on the vibrational frequency for non-axisymmetric vibrations ($n=1$) for the third mode of frequency

(k) from 3.6 to 20, the non local parameter (e_0a) influences on the maximum values of second mode (ω_1). At $e_0a = 0nm$, $e_0a = 1nm$, $e_0a = 2nm$, $e_0a = 3nm$ and $e_0a = 4nm$, ω_2 has maixium value 1×10^{-1} , 4×10^{-2} , 10^{-2} , 5×10^{-4} , and 2×10^{-4} at the value of wave vector $k = 20$ respectively. Fig. 2, also shows that for the second mode of frequency, there is negligiale dufference between the classical and non local results when the wave vector (k) obtain the values between 0 and 4. But there exist an outstading difference between clasical and non local results when the values of wave vector between 4.2 and 20. The non local parameter (e_0a) influences on the maximum values the of second mode (ω_2). At $e_0a = 0nm$, $e_0a = 1nm$, $e_0a = 2nm$, $e_0a = 3nm$ and $e_0a = 4nm$, ω_2 has maixium value 10^{-3} , 10^{-4} , 5×10^{-5} , 2×10^{-5} , and 9×10^{-6} at the value of wave vector $k = 20$ respectively. Fig. 3, shows that for the third mode of the frequency, there is a slight difference between classical and non-local result when the wave vectors (k) obtains the values between 0 to 3. But there is huge difference between the classical and nonlocal results when the wave vector (k) obtains values between 3.1 to 20. The non local parameter (e_0a) influences on the maximum values of second mode (ω_2). At $e_0a = 0nm$, $e_0a = 1nm$, $e_0a = 2nm$, $e_0a = 3nm$ and $e_0a = 4nm$, ω_2 has maixium value

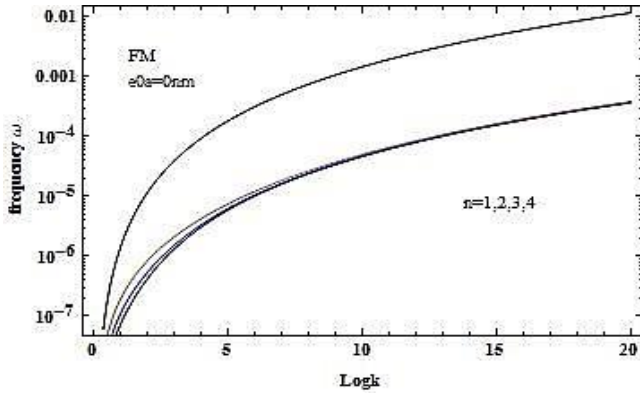


Fig. 4 Effects of half circumferential wave number n on vibrational frequency for nonlocal case $e_0a = 0nm$ of ω_1

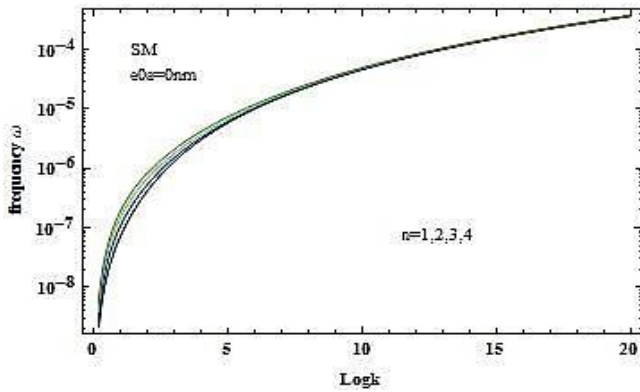


Fig. 5 Effects of half circumferential wave number n on vibrational frequency for nonlocal case $e_0a = 0nm$ of ω_2

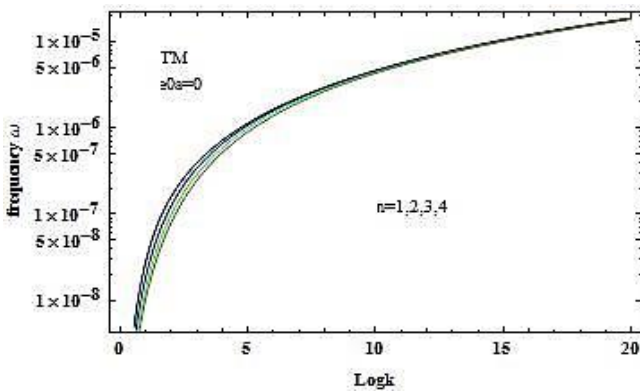


Fig. 6 Effects of half circumferential wave number n on vibrational frequency for nonlocal case $e_0a = 0nm$ of (ω_3) (Third mode)

3×10^{-5} , 5×10^{-6} , 1×10^{-6} , 5×10^{-7} and 3×10^{-7} at the $k = 20$ respectively. Fig. 4, shows effects different values of n (half circumferential wave number) on the vibrational frequency for nonlocal case $e_0a = 0nm$ of the first mode. effects of different values of n (half circumferential wave number) on the vibrational frequency for nonlocal case $e_0a = 0nm$ of the second mode as shown in Fig. 5. Fig. 6, depicts the effect of different values of n (half circumferential wave number) on the vibrational frequency for nonlocal case $e_0a = 0nm$ of the third mode. The effects of different values of n (half circumferential

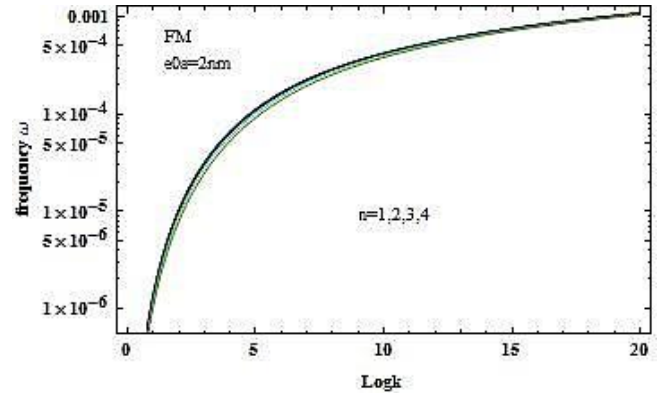


Fig. 7 Effects of half circumferential wave number n on vibrational frequency for non-local case $e_0a = 2nm$ of the first mode

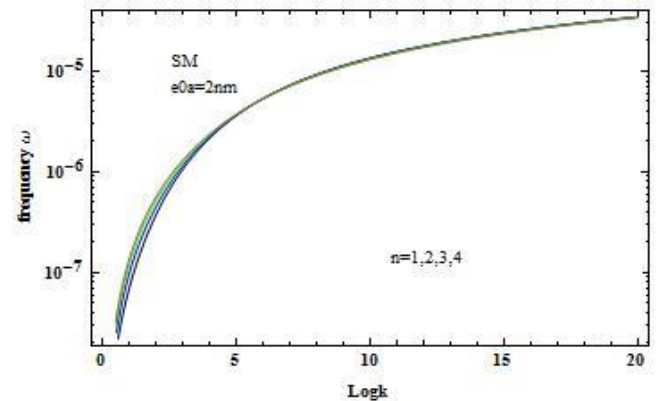


Fig. 8 Effects of half circumferential wave number n on vibrational frequency for nonlocal case $e_0a = 2nm$ of the (ω_2) (Second mode)

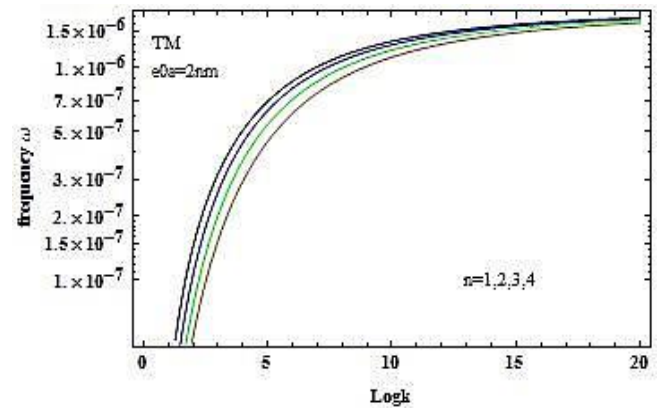


Fig. 9 Effects of half circumferential wave number n on vibrational frequency for nonlocal case $e_0a = 2nm$ of the ω_3 (Third mode)

wave number) on the vibrational frequency for nonlocal case $e_0a = 2nm$ of the first mode as shown in Fig 7. Fig. 8, show the effects of different values of n (half circumferential wave number) on the vibrational frequency for nonlocal case $e_0a = 2nm$ of the second mode. Fig. 9, show the effects of the different values of n (half circumferential wave number) on the vibrational frequency for nonlocal case $e_0a = 2nm$ of the third mode.

5. Conclusion

MTs are modeled as non-local orthotropic elastic shell model and “wave propagation approach” is used to solve the obtained model. We observed that due to non-local effects the frequency of vibration is increased with the increase in wave vector. The effects of small scale parameters can increase vibrational frequencies of the protein microtubules, and cannot be overlooked in analysis of vibrating phenomena. We checked that the non-local parameter (e_0a) affected the first mode (ω_1) more than second mode (ω_2) and third mode (ω_3) of vibration. In the future, we may try to check the effects of surrounding medium and thermal stress on the vibration of MTs taking into account its non-linearity

Declaration of Conflicting Interests

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