

## Hygro-thermal effects on wave dispersion responses of magnetostrictive sandwich nanoplates

Farzad Ebrahimi <sup>\*1</sup>, Ali Dabbagh <sup>1</sup>, Francesco Tornabene <sup>2</sup> and Omer Civalek <sup>3</sup>

<sup>1</sup> Department of Mechanical Engineering, Faculty of Engineering, Imam Khomeini International University, Qazvin, Iran

<sup>2</sup> Department of Civil, Chemical, Environmental, and Materials Engineering, University of Bologna, Bologna, Italy

<sup>3</sup> Akdeniz University, Engineering Faculty, Civil Engineering Department, Division of Mechanics, 07058 Antalya, Turkey

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**Abstract.** In this paper, a classical plate model is utilized to formulate the wave propagation problem of magnetostrictive sandwich nanoplates (MSNPs) while subjected to hygrothermal loading with respect to the scale effects. Herein, magnetostriction effect is considered and controlled on the basis of a feedback control system. The nanoplate is supposed to be embedded on a visco-Pasternak substrate. The kinematic relations are derived based on the Kirchhoff plate theory; also, combining these obtained equations with Hamilton's principle, the local equations of motion are achieved. According to a nonlocal strain gradient theory (NSGT), the small scale influences are covered precisely by introducing two scale coefficients. Afterwards, the nonlocal governing equations can be derived coupling the local equations with those of the NSGT. Applying an analytical solution, the wave frequency and phase velocity of propagated waves can be gathered solving an eigenvalue problem. On the other hand, accuracy and efficiency of presented model is verified by setting a comparison between the obtained results with those of previous published researches. Effects of different variants are plotted in some figures and the highlights are discussed in detail.

**Keywords:** wave propagation; magnetostrictive materials; nonlocal strain gradient theory (NSGT); sandwich nanoplates; hygro-thermal environments

### 1. Introduction

As a matter of fact, a giant state of art in smart materials is devoted to the magnetostrictive materials without any exaggeration at all. Actually, these materials are able to exchange magnetic motivations to elastic responses which can be elongation or contraction. It is worth mention that this mentioned magneto-elastic mutual interaction appears as the structure is subjected to a magnetic field. Due to these capabilities, the structures in which possess the magnetostriction effect are excellent candidates for the purpose of being utilized as ultrasonic generators, ultrasonic receivers and echo detectors (Yifeng *et al.* 2013). To this reasons, some of the researchers performed various mechanical analyses on the structures made of such smart materials. Reddy and Barbosa (2000) tried to show how can prevent from vibration in a magnetostrictive composite beam employing both first and third order shear deformation beam models. After about a decade, Hong (2009) presented a new paper dealing with the thermo-mechanical transient dynamic responses of laminated composite plates. The free vibration problem of laminated composite curved beams including magnetostrictive layers is solved by Bayat *et al.* (2015).

Besides, in the recent years, a considerable increasing

use of nanosize structures in various nano-electro-mechanical-systems (NEMSs) is observed which makes it obligatory for designers to earn reliable information about the mechanical characteristics of structures in small scales. Hence, classical continuum theories must be replaced by size-dependent continuum theories to take into consider the scale effects once analyzing the nano-mechanical responses of elements. To this reason, Eringen (1983) presented the first nonlocal continuum theory, called nonlocal elasticity theory (NET). Hitherto, lots of authors performed their researches on the nanostructures in the framework of this theory. Natarajan *et al.* (2012) could show the scale effects while studying the dynamic behaviors of small composite plates based on the NET. Also, the NE is utilized by Hosseini-Hashemi *et al.* (2013) to survey the vibration problem of nanosize plates with respect to shear deformation applying a first-order shear deformable plate model. Moreover, Yan *et al.* (2015) captured the small scale influences in the framework of the NE while investigating the bending characteristics of nano scale beams and plates. Ebrahimi *et al.* (2015) used the NET to analyze the thermo-elastic vibrational characteristics of nanobeams. The effect of exact position of the neutral axis of a composite nanobeam on the dynamic responses of this nanostructure is considered by Ahouel *et al.* (2016) within the framework of the nonlocal hypothesis of Eringen. In addition, Ebrahimi and Hosseini (2016) studied the vibration problem of viscoelastic double-layered nanoplates once both thermal and small scale effects are considered. Moreover, the stability behaviors of composite nanobeams in hygro-thermal environments and under an induced magnetic force

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\*Corresponding author, Associate Professor,  
E-mail: febrahimi@gmail.com

are investigated by Ebrahimi and Barati (2016) according to the NET. The effects of small scale are considered by Bellifa *et al.* (2017) in an article dealing with the nonlocal post-buckling responses of nanobeams. Besides, (Mouffoki *et al.* 2017) have probed the hygro-thermally influenced dynamic characteristics of nanobeams via a new two-variable kinematic theorem. Hamza-Cherif *et al.* (2018) utilized a powerful numerical method for the goal of analyzing the thermally affected vibrational behaviors of carbon nanotubes simulated with a nanobeam. On the other hand, some of the scientific works deal with the efficiency of the size-dependent continuum theories. In fact, in the some of the published researches it is proven that the NET of Eringen is not able to cover the scale effects completely (Fleck and Hutchinson 1993, Stölken and Evans 1998). Actually, according to the NE, nonlocality results in a decreasing effect on the mechanical responses of nanosize structures; whereas, the scale influences cannot be summarized in the framework of this decreasing impact. In other words, there is a totally different increasing effect which is predicted in the strain gradient elasticity which is neglected in the NE. Lately, Lim *et al.* (2015) tried to combine these effects to present a general theory which is efficient enough to take into account for both of the mentioned effects. In this new theory, named nonlocal strain gradient theory (NSGT), two scale coefficients are introduced to capture both diminishing and strengthening impacts of small size. Li and Hu (2015) tried to highlight the effects of strain gradient stress field on the buckling responses of geometrically nonlinear nanobeams. The buckling characteristics of NSG based orthotropic graphene sheets are considered by Farajpour *et al.* (2016). In addition, Ebrahimi and Barati (2018b) have investigated the dynamic behaviors of graphene sheets embedded on an orthotropic elastic medium under hygro-thermo-magnetic loadings via a NSGT. Besides, Ebrahimi *et al.* (2017) presented a NSGT for wave propagation analysis of compositionally graded nanobeams in thermal environments. Also, Ebrahimi and Barati (2017) surveyed the thermo-magneto-elastic dynamic responses of axially graded nanosize beams based on the NSGT. In another paper, Li *et al.* (2018b) have surveyed the nonlinear natural frequency behaviors of a porous nanobeam by the means of the NSGT by considering the effect of existence of an imperfection. The dynamic responses of double-layered viscoelastic graphene sheets are reviewed by Ebrahimi and Barati (2018a) on the basis of a NSGT coupled with a refined higher-order shear deformable two-variable plate theory. Lately, Karami *et al.* (2019) probed the effects of hygro-thermal environments on the dispersion characteristics of a porous nanobeam whenever the effects of magnetic field on the nanostructure are included.

In the recent years, a remarkable interest in the science society is allocated to the static and dynamic analyses of tiny structures, whereas, the wave propagation problem of such small scale elements has never been mentioned a lot. Only a few researches can be found about the dispersion characteristics of waves propagating through an element. Ebrahimi and his co-workers performed their researches on the wave dispersion answers of composite nanosize beams

and plates under various thermo-electro-magneto-elastic loadings via different nonlocal continuum theories (Ebrahimi *et al.* 2016a, b, c, 2018a, b, Ebrahimi and Dabbagh 2017a, b, c, d, 2018a, b, c, e, f). Ghorbanpour Arani *et al.* (2017) studied the wave propagation behaviors of viscoelastic sandwich nanoplates with respect to surface effects and smart properties of nanoplate. In another attempt, Xiao *et al.* (2017) examined the wave propagation characteristics of monolayer graphene sheets via a NSG based plate model. Karami *et al.* (2018) employed the NSGT incorporated with the higher-order kinematic hypothesis of doubly-curved nanoshells in order to study the wave propagation problem of such nanostructures. Ebrahimi and Dabbagh (2018g) tried to examine the wave propagation problem of a magneto-strictive sandwich nanoplate (MSNP) incorporating nonlocal stress-strain gradient elasticity theory. In the most recent research in this area, Ebrahimi and Dabbagh (2018d) probed the wave propagation problem of a MSNP in thermal environments subjected to an external induced magnetic force. It can be figured out that the issue of analyzing the wave dispersion problem in the nanosize MS structures must be investigated more than before. Indeed, only two papers can be found dealing with the wave propagation features of MSNPs. Obviously, it can be easily found that the wave dispersion responses of MSNPs once subjected to hygro-thermal loadings has never been surveyed up to now.

Present work is going to solve this lack of information about the hygro-thermo-elastic wave propagation characteristics of MSNPs employing a NSG plate model. The displacement fields of the SNP are assumed to be derived applying the kinematic relations of classical plate theory (CPT). Furthermore, the system is considered to be rested on a viscoelastic medium which is made of Winkler, Pasternak and damping coefficients. According to a Hamiltonian approach, the Euler-Lagrange equations are reached. The magnetostriction impact is controlled via a feedback control system. The nonlocal governing equations are solved analytically and the wave frequency and phase velocity values are obtained. At the end, effect of different parameters is highlighted in some separate diagrams.

## 2. Theory and formulation

### 2.1 Stress-strain relations for magnetostrictive materials

The relation between stress, strain and magnetic field can be explained for magnetostrictive materials as Ebrahimi and Dabbagh (2018g)

$$\sigma_{xx} = C_{11}\epsilon_{xx} + C_{12}\epsilon_{yy} - e_{31}H_z \quad (1)$$

$$\sigma_{yy} = C_{21}\epsilon_{xx} + C_{22}\epsilon_{yy} - e_{32}H_z \quad (2)$$

$$\sigma_{xy} = C_{66}\epsilon_{xy} - e_{34}H_z \quad (3)$$

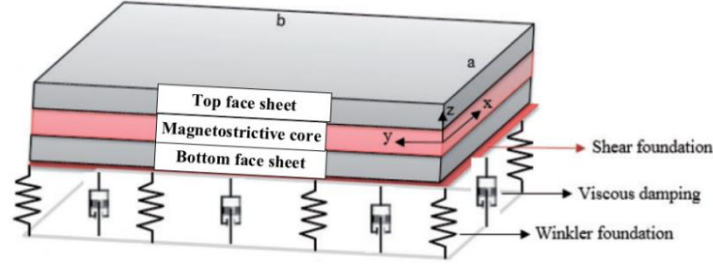


Fig. 1 Schematic of a MSNP rested on a visco-Pasternak medium

where  $\sigma_{ij}$ ,  $\varepsilon_{ij}$ ,  $e_{ij}$  and  $H_z$  denote stress, strain, magnetostrictive constant and magnetic field, respectively.

In addition,  $C_{ij}$  stand for the elastic stiffness coefficient which are defined as Ebrahimi and Dabbagh (2018g)

$$C_{11} = C_{22} = \frac{E_c}{1-\nu_c^2}, \quad C_{12} = \frac{\nu_c E_c}{1-\nu_c^2}, \quad C_{66} = \frac{E_c}{2(1+\nu_c)} \quad (4)$$

in which  $E_c$  and  $\nu_c$  denote the modulus of elasticity and Poisson's ratio of the core, respectively. Here, it is interesting to inform about a reciprocal magneto-elastic interaction which is available in magnetostrictive materials. Indeed, such materials are able to tolerate elongation or compression once subjected to a magnetic field. This phenomenon is reported to be induced due to the reorientation of the atomic magnetic moments (Reddy and Barbosa 2000). Moreover, on the basis of a feedback control system it is tried to account for the intensity of magnetic field for the goal of capturing the magnetization effect in MSCNP. According to this model as illustrated in Fig. 1, the magnetostrictive coefficients can be easily formulated in the following (Ebrahimi and Dabbagh 2018g)

$$e_{31} = \tilde{e}_{31} \cos^2 \psi + \tilde{e}_{32} \sin^2 \psi \quad (5)$$

$$e_{32} = \tilde{e}_{32} \cos^2 \psi + \tilde{e}_{31} \sin^2 \psi \quad (6)$$

in which  $\psi$  denotes the angle that the magnetic anisotropy might be induced. In the present research, a unidirectional magnetic field along  $z$  axis is utilized to be applied to the MSCNP to account for the magnetization effects. Now, the magnetic field intensity can be expressed as follows (Ebrahimi and Dabbagh 2018g)

$$H_z = K_c I(x, y, t) = K_c C(t) \frac{\partial w(x, y, t)}{\partial t} \quad (7)$$

in which  $K_c$  is the coil constant which is a function of the coil characteristics such as its width, radius and number of turns of the coil. Actually, coil constant can be formulated as (Ebrahimi and Dabbagh 2018g, Reddy and Barbosa 2000)

$$K_c = \frac{n_c}{\sqrt{b_c^2 + 4r_c^2}} \quad (8)$$

where  $n_c$ ,  $b_c$  and  $r_c$  are related to turns of coil, coil width and coil radius, respectively. Besides,  $I(t)$  and  $C(t)$  stand for the

coil current and control gain, respectively. It shall be considered that the control gain is presumed to be constant. Finally,  $K_c C(t)$  is the velocity feedback gain.

On the other hand, the stress-strain relations of the top and bottom layers are going to be explained. As same as the central core, here, for the upper and lower face sheets the above equations are employed (Ebrahimi and Dabbagh 2018g)

$$\sigma_{xx} = Q_{11} \varepsilon_{xx} + Q_{12} \varepsilon_{yy} \quad (9)$$

$$\sigma_{yy} = Q_{21} \varepsilon_{xx} + Q_{22} \varepsilon_{yy} \quad (10)$$

$$\sigma_{xy} = Q_{66} \varepsilon_{xy} \quad (11)$$

where

$$Q_{11} = \frac{E_f}{1-\nu_f^2}, \quad Q_{12} = \frac{\nu_f E_f}{1-\nu_f^2}, \quad Q_{22} = \frac{E_f}{1-\nu_f^2}, \quad Q_{66} = \frac{E_f}{2(1+\nu_f)} \quad (12)$$

in which  $E_f$  is Young's modulus and  $\nu_f$  is Poisson's ratios.

## 2.2 Kinematic relations

The equations of motion for the plates are modeled in the present research according to the classical plate theory in the following form (Ebrahimi and Dabbagh 2018g)

$$u_x(x, y, z, t) = u(x, y, t) - z \frac{\partial w(x, y, t)}{\partial x} \quad (13)$$

$$u_y(x, y, z, t) = v(x, y, t) - z \frac{\partial w(x, y, t)}{\partial y} \quad (14)$$

$$u_z(x, y, z, t) = w(x, y, t) \quad (15)$$

in which,  $u$ ,  $v$  and  $w$  correspond with the displacement components in the  $x$ ,  $y$  and  $z$  directions, respectively. In this study, the influences of shear stress and strain are ignored, however, one can reach more information about this issue by reading complementary references (Abdelaziz *et al.* 2017, Beldjelili *et al.* 2016). Besides, whenever the effects of the thickness stretching are of importance, the researchers are advised to employ the quasi-3D models instead of the conventional ones. One can obtain more

detailed data by looking at the references in this field (Zaoui *et al.* 2019, Abualnour *et al.* 2018). Therefore, the nonzero strains of the core plate can be defined as (Ebrahimi and Dabbagh 2018g)

$$\begin{Bmatrix} \varepsilon_{xx}^{core} \\ \varepsilon_{yy}^{core} \\ \varepsilon_{xy}^{core} \end{Bmatrix} = \begin{Bmatrix} \varepsilon_{xx}^{core} \\ \varepsilon_{yy}^{core} \\ \varepsilon_{xy}^{core} \end{Bmatrix} + z \begin{Bmatrix} \kappa_{xx}^{core} \\ \kappa_{yy}^{core} \\ \kappa_{xy}^{core} \end{Bmatrix} \quad (16)$$

where

$$\begin{Bmatrix} \varepsilon_{xx}^{core} \\ \varepsilon_{yy}^{core} \\ \varepsilon_{xy}^{core} \end{Bmatrix} = \begin{Bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{Bmatrix}, \quad \begin{Bmatrix} \kappa_{xx}^{core} \\ \kappa_{yy}^{core} \\ \kappa_{xy}^{core} \end{Bmatrix} = \begin{Bmatrix} -\frac{\partial^2 w}{\partial x^2} \\ -\frac{\partial^2 w}{\partial y^2} \\ -2\frac{\partial^2 w}{\partial x \partial y} \end{Bmatrix} \quad (17)$$

Moreover, for the face sheets the same relations can be developed (Ebrahimi and Dabbagh 2018g)

$$\begin{Bmatrix} \varepsilon_{xx}^{face} \\ \varepsilon_{yy}^{face} \\ \varepsilon_{xy}^{face} \end{Bmatrix} = \begin{Bmatrix} \varepsilon_{xx}^{face} \\ \varepsilon_{yy}^{face} \\ \varepsilon_{xy}^{face} \end{Bmatrix} + z \begin{Bmatrix} \kappa_{xx}^{face} \\ \kappa_{yy}^{face} \\ \kappa_{xy}^{face} \end{Bmatrix} \quad (18)$$

where

$$\begin{Bmatrix} \varepsilon_{xx}^{face} \\ \varepsilon_{yy}^{face} \\ \varepsilon_{xy}^{face} \end{Bmatrix} = \begin{Bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{Bmatrix}, \quad \begin{Bmatrix} \kappa_{xx}^{face} \\ \kappa_{yy}^{face} \\ \kappa_{xy}^{face} \end{Bmatrix} = \begin{Bmatrix} -\frac{\partial^2 w}{\partial x^2} \\ -\frac{\partial^2 w}{\partial y^2} \\ -2\frac{\partial^2 w}{\partial x \partial y} \end{Bmatrix} \quad (19)$$

Also, it is interesting that in some of the papers, researchers select a new version of the classical beam or plate models that are modified in a way that can capture the effect of shear deformation (Kaci *et al.* 2018, Zidi *et al.* 2017, Houari *et al.* 2016). Now, according to the Hamilton's principle the Euler-Lagrange equations of the MSP as follows (Ebrahimi and Barati 2018b)

$$\int_0^t \delta(U - T + V) dt = 0 \quad (20)$$

where  $U$ ,  $T$  and  $V$  account for strain energy, kinetic energy and work done by external forces, respectively. Now, the variation of strain energy for core and face sheets can be formulated as (Ebrahimi and Dabbagh 2018g)

$$\delta U^{core} = \int_0^a \left[ N_{xx}^{core} (\delta \varepsilon_{xx}^{core}) + M_{xx}^{core} (\delta \kappa_{xx}^{core}) + N_{yy}^{core} (\delta \varepsilon_{yy}^{core}) + M_{yy}^{core} (\delta \kappa_{yy}^{core}) + N_{xy}^{core} (\delta \varepsilon_{xy}^{core}) + M_{xy}^{core} (\delta \kappa_{xy}^{core}) \right] dx \quad (21)$$

$$\delta U^{face} = \int_0^a \left[ N_{xx}^{face} (\delta \varepsilon_{xx}^{face}) + M_{xx}^{face} (\delta \kappa_{xx}^{face}) + N_{yy}^{face} (\delta \varepsilon_{yy}^{face}) + M_{yy}^{face} (\delta \kappa_{yy}^{face}) + N_{xy}^{face} (\delta \varepsilon_{xy}^{face}) + M_{xy}^{face} (\delta \kappa_{xy}^{face}) \right] dx \quad (22)$$

in above equation, the axial forces and bending moments can be defined as (Ebrahimi and Dabbagh 2018g)

$$(N_{ij}^{core}, M_{ij}^{core}) = \int_{-\frac{h_c}{2}}^{\frac{h_c}{2}} \int_0^b (1, z) \sigma_{ij}^{core} dy dz, \quad (i, j = x, y) \quad (23)$$

$$(N_{ij}^{face}, M_{ij}^{face}) = \left[ \int_{\frac{(h_c+h_f)}{2}}^{\frac{h_c}{2}} \int_0^b (1, z) \sigma_{ij}^{face} dy dz + \int_{\frac{h_c}{2}}^{\frac{(h_c+h_f)}{2}} \int_0^b (1, z) \sigma_{ij}^{face} dy dz \right], \quad (i, j = x, y) \quad (24)$$

Also, the variation of kinetic energy can be expressed as follows (Ebrahimi and Dabbagh 2018g)

$$\delta T = \int_0^L \left( I_0 \left( \frac{\partial u}{\partial t} \frac{\partial \delta u}{\partial t} + \frac{\partial v}{\partial t} \frac{\partial \delta v}{\partial t} + \frac{\partial w}{\partial t} \frac{\partial \delta w}{\partial t} \right) + I_2 \left( \frac{\partial^2 w}{\partial x \partial t} \frac{\partial^2 \delta w}{\partial x \partial t} + \frac{\partial^2 w}{\partial y \partial t} \frac{\partial^2 \delta w}{\partial y \partial t} \right) \right) dx \quad (25)$$

in which, the mass moments of inertia are defined as (Ebrahimi and Dabbagh 2018g)

$$I_0 = \int_{-\frac{h_c}{2}}^{\frac{h_c}{2}} \int_0^b \rho_c dy dz + \left[ \int_{\frac{(h_c+h_f)}{2}}^{\frac{h_c}{2}} \int_0^b \rho_f dy dz + \int_{\frac{h_c}{2}}^{\frac{(h_c+h_f)}{2}} \int_0^b \rho_f dy dz \right] \quad (26)$$

$$I_2 = \int_{-\frac{h_c}{2}}^{\frac{h_c}{2}} \int_0^b \rho_c z^2 dy dz + \left[ \int_{\frac{(h_c+h_f)}{2}}^{\frac{h_c}{2}} \int_0^b \rho_f z^2 dy dz + \int_{\frac{h_c}{2}}^{\frac{(h_c+h_f)}{2}} \int_0^b \rho_f z^2 dy dz \right] \quad (27)$$

where  $\rho_c$  and  $\rho_f$  correspond with the density of the core plate and face sheets, respectively. Now, the variation of the external work done by external forces can be written in the following form (Ebrahimi and Dabbagh 2018g, Ebrahimi and Barati 2018b)

$$\delta V = \int_0^L \left[ -k_w \delta w + k_p \left( \frac{\partial^2 \delta w}{\partial x^2} + \frac{\partial^2 \delta w}{\partial y^2} \right) - c_d \frac{\partial w}{\partial t} - (N^T + N^H) \left( \frac{\partial^2 \delta w}{\partial x^2} + \frac{\partial^2 \delta w}{\partial y^2} \right) \right] dx \quad (28)$$

where  $k_w$ ,  $k_p$  and  $c_d$  are Winkler, Pasternak and damping coefficients of elastic medium, respectively. Also,  $N^T$  and  $N^H$  stand for the applied loads due to the temperature rise and moisture concentration, respectively. Herein, once Eqs. (21), (22), (25) and (28) are substituted in Eq. (20) and the coefficients of  $\delta u$ ,  $\delta v$  and  $\delta w$  are set to zero, the Euler-Lagrange equations of MSCPs can be written as follows (Ebrahimi and Dabbagh 2018g, Ebrahimi and Barati 2018b)

$$\frac{\partial N_{xx}}{\partial x} + \frac{\partial N_{xy}}{\partial y} = I_0 \frac{\partial^2 u}{\partial t^2} + \frac{e_{31} K_c C(t) h_c}{E_c} \frac{\partial^2 w}{\partial x \partial t} \quad (29)$$

$$\frac{\partial N_{xy}}{\partial x} + \frac{\partial N_{yy}}{\partial y} = I_0 \frac{\partial^2 v}{\partial t^2} + \frac{e_{32} K_c C(t) h_c}{E_c} \frac{\partial^2 w}{\partial y \partial t} \quad (30)$$

$$\begin{aligned} & \frac{\partial^2 M_{xx}}{\partial x^2} + 2 \frac{\partial^2 M_{xy}}{\partial x \partial y} + \frac{\partial^2 M_{yy}}{\partial y^2} - (N^T + N^H) \nabla^2 w \\ & = I_0 \frac{\partial^2 w}{\partial t^2} - I_2 \nabla^2 \frac{\partial^2 w}{\partial t^2} + k_w w - k_p \nabla^2 w + c_d \frac{\partial w}{\partial t} \\ & + \frac{e_{31} K_c C(t) h_c}{E_c} \frac{\partial^2 u}{\partial x \partial t} + \frac{e_{32} K_c C(t) h_c}{E_c} \frac{\partial^2 v}{\partial y \partial t} \end{aligned} \quad (31)$$

### 2.3 Nonlocal strain gradient elasticity

According to this theory, two nonlocal and length scale parameters should be included in an efficient theory which is powerful enough to take into consider the scale influences. So, the initial form of this theory can be expressed as (Ebrahimi *et al.* 2016a)

$$\sigma_{ij} = \sigma_{ij}^{(0)} - \frac{d\sigma_{ij}^{(1)}}{dx} \quad (32)$$

where  $\sigma_{ij}^{(0)}$  and  $\sigma_{ij}^{(1)}$  are classical and higher order stresses which are related to the strain ( $\varepsilon_{xx}$ ) and strain gradient ( $\varepsilon_{xx,x}$ ), respectively. These stresses can be formulated as Ebrahimi *et al.* (2016a)

$$\sigma_{ij}^{(0)} = \int_0^L C_{ijkl} \alpha_0(x, x', e_0 a) \varepsilon'_{kl}(x') dx' \quad (33)$$

$$\sigma_{ij}^{(1)} = l^2 \int_0^L C_{ijkl} \alpha_1(x, x', e_1 a) \varepsilon'_{kl,x}(x') dx' \quad (34)$$

in which  $C_{ijkl}$  is the elastic coefficient;  $e_0 a$  and  $e_1 a$  are introduced to account for the nonlocality effects. Also,  $l$  captures the strain gradient effects. Once the nonlocal kernel functions  $\alpha_0(x, x', e_0 a)$  and  $\alpha_1(x, x', e_1 a)$  satisfy the developed conditions, the constitutive relation of nonlocal strain gradient theory can be expressed as below (Ebrahimi *et al.* 2016a)

$$\begin{aligned} & (1 - (e_1 a)^2 \nabla^2) (1 - (e_0 a)^2 \nabla^2) \sigma_{ij} \\ & = C_{ijkl} (1 - (e_1 a)^2 \nabla^2) \varepsilon_{kl} - C_{ijkl} l^2 (1 - (e_0 a)^2 \nabla^2) \nabla^2 \varepsilon_{kl} \end{aligned} \quad (35)$$

in which  $\nabla^2$  denotes the Laplacian operator. Considering  $e_1$

$= e_0 = e$ , the general constitutive relation in Eq. (35) can be stated as follows with respect to hygrothermal influences (Ebrahimi and Barati 2018b, Ebrahimi *et al.* 2016a)

$$(1 - \mu^2 \nabla^2) \sigma_{ij} = C_{ijkl} (1 - \lambda^2 \nabla^2) (\varepsilon_{kl} - \alpha_{ij} T - \beta_{ij} C) \quad (36)$$

where  $\mu = ea$  and  $\lambda = l$  are nonlocal and length scale parameters, respectively. Furthermore,  $\alpha_{ij}$  and  $\beta_{ij}$  denote the thermal and moisture expansion coefficients, respectively; also,  $T$  and  $C$  stand for temperature in terms of Kelvin degrees and moisture concentration, respectively. It is worth mentioning that in some cases, the effect of the Poisson's ratio and the structures' thickness on the constitutive behaviors of nanostructures are studied (Tang *et al.* 2019, Li *et al.* 2018a). Now, the final nonlocal relations of normal forces and bending moments can be achieved by integrating from Eq. (36) over the cross section area of the plate as (Ebrahimi and Dabbagh 2018g)

$$\begin{aligned} & (1 - \mu^2 \nabla^2) \begin{bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \end{bmatrix} \\ & = (1 - \lambda^2 \nabla^2) \begin{bmatrix} A_{11} & A_{12} & 0 \\ A_{21} & A_{22} & 0 \\ 0 & 0 & A_{66} \end{bmatrix} \begin{bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{bmatrix} + \begin{bmatrix} B_{11} & B_{12} & 0 \\ B_{21} & B_{22} & 0 \\ 0 & 0 & B_{66} \end{bmatrix} \begin{bmatrix} -\frac{\partial^2 w}{\partial x^2} \\ -\frac{\partial^2 w}{\partial y^2} \\ -2\frac{\partial^2 w}{\partial x \partial y} \end{bmatrix} \end{aligned} \quad (38)$$

$$\begin{aligned} & (1 - \mu^2 \nabla^2) \begin{bmatrix} M_{xx} \\ M_{yy} \\ M_{xy} \end{bmatrix} \\ & = (1 - \lambda^2 \nabla^2) \begin{bmatrix} B_{11} & B_{12} & 0 \\ B_{21} & B_{22} & 0 \\ 0 & 0 & B_{66} \end{bmatrix} \begin{bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{bmatrix} + \begin{bmatrix} D_{11} & D_{12} & 0 \\ D_{21} & D_{22} & 0 \\ 0 & 0 & D_{66} \end{bmatrix} \begin{bmatrix} -\frac{\partial^2 w}{\partial x^2} \\ -\frac{\partial^2 w}{\partial y^2} \\ -2\frac{\partial^2 w}{\partial x \partial y} \end{bmatrix} \end{aligned} \quad (37)$$

where

$$\begin{aligned} & \begin{bmatrix} A_{11} & B_{11} & D_{11} \\ A_{22} & B_{22} & D_{22} \\ A_{12} & B_{12} & D_{12} \\ A_{66} & B_{66} & D_{66} \end{bmatrix} = \int_{-\frac{h_c}{2}}^{\frac{h_c}{2}} \int_0^b \begin{bmatrix} 1 & z & z^2 \end{bmatrix} \begin{bmatrix} C_{11} \\ C_{22} \\ C_{12} \\ C_{66} \end{bmatrix} dy dz \\ & + \int_{-\frac{(h_c+h_f)}{2}}^{\frac{h_c}{2}} \int_0^b \begin{bmatrix} 1 & z & z^2 \end{bmatrix} \begin{bmatrix} Q_{11} \\ Q_{22} \\ Q_{12} \\ Q_{66} \end{bmatrix} dy dz \end{aligned} \quad (39)$$

$$S_{21} = S_{12}, \quad (S = A, B, D)$$

Now, the final governing equations of MSNPs can be achieved by substituting Eqs. (37) and (38) in Eqs. (29)-(31)

$$(1 - \lambda^2 \nabla^2) \left( A_{11} \frac{\partial^2 u}{\partial x^2} + (A_{12} + A_{66}) \frac{\partial^2 v}{\partial x \partial y} + A_{66} \frac{\partial^2 u}{\partial y^2} \right) \quad (40)$$

$$-B_{11} \frac{\partial^3 w}{\partial x^3} - (B_{12} + 2B_{66}) \frac{\partial^3 w}{\partial x \partial y^2} \Bigg) + (1 - \mu^2 \nabla^2) \left( -I_0 \frac{\partial^2 u}{\partial t^2} - \frac{e_{31} K_c C(t) h_c}{E_c} \frac{\partial^2 w}{\partial x \partial t} \right) = 0 \quad (40)$$

$$(1 - \lambda^2 \nabla^2) \left( A_{22} \frac{\partial^2 v}{\partial y^2} + (A_{12} + A_{66}) \frac{\partial^2 u}{\partial x \partial y} + A_{66} \frac{\partial^2 v}{\partial x^2} - B_{22} \frac{\partial^3 w}{\partial y^3} - (B_{12} + 2B_{66}) \frac{\partial^3 w}{\partial x^2 \partial y} \right) + (1 - \mu^2 \nabla^2) \left( -I_0 \frac{\partial^2 v}{\partial t^2} - \frac{e_{32} K_c C(t) h_c}{E_c} \frac{\partial^2 w}{\partial y \partial t} \right) = 0 \quad (41)$$

$$(1 - \lambda^2 \nabla^2) \left( B_{11} \frac{\partial^3 u}{\partial x^3} + (B_{12} + 2B_{66}) \frac{\partial^3 u}{\partial x \partial y^2} + B_{22} \frac{\partial^3 v}{\partial y^3} + (B_{12} + 2B_{66}) \frac{\partial^3 v}{\partial x^2 \partial y} - D_{11} \frac{\partial^4 w}{\partial x^4} - 2(D_{12} + 2D_{66}) \frac{\partial^4 w}{\partial x^2 \partial y^2} - D_{22} \frac{\partial^4 w}{\partial y^4} \right) + (1 - \mu^2 \nabla^2) \left( -I_0 \frac{\partial^2 w}{\partial t^2} + I_2 \nabla^2 \frac{\partial^2 w}{\partial t^2} - k_w w + k_p \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) - c_d \frac{\partial w}{\partial t} - (N^T + N^H) \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) + \frac{e_{31} K_c C(t) h_c}{E_c} \frac{\partial^2 u}{\partial x \partial t} + \frac{e_{32} K_c C(t) h_c}{E_c} \frac{\partial^2 v}{\partial y \partial t} \right) = 0 \quad (42)$$

in which

$$\begin{cases} N^T = \int_{-\frac{h_c}{2}}^{\frac{h_c}{2}} \int_0^b \frac{E_c}{1-\nu_c} \alpha_c \Delta T dy dz + \int_{-\frac{h_f}{2}}^{\frac{h_f}{2}} \int_0^b \frac{E_f}{1-\nu_f} \alpha_f \Delta T dy dz + \int_{-\frac{h_c+h_f}{2}}^{\frac{h_c+h_f}{2}} \int_0^b \frac{E_f}{1-\nu_f} \alpha_f \Delta T dy dz \\ N^H = \int_{-\frac{h_c}{2}}^{\frac{h_c}{2}} \int_0^b \frac{E_c}{1-\nu_c} \beta_c \Delta C dy dz + \int_{-\frac{h_f}{2}}^{\frac{h_f}{2}} \int_0^b \frac{E_f}{1-\nu_f} \beta_f \Delta C dy dz + \int_{-\frac{h_c+h_f}{2}}^{\frac{h_c+h_f}{2}} \int_0^b \frac{E_f}{1-\nu_f} \beta_f \Delta C dy dz \end{cases} \quad (43)$$

In above relation,  $\alpha_c$ ,  $\alpha_f$ ,  $\beta_c$ , and  $\beta_f$  correspond with the thermal and moisture expansion coefficients of core and face sheets, respectively.

## 2.4 Analytical solution

Here, an analytical solution method is applied to solve the governing equations. The details of the wave propagation analysis can be more realized by studying the complementary references about the wave propagation responses of composite plates (Yahia *et al.* 2015). The displacements are supposed to be as

$$\begin{cases} u \\ v \\ w \end{cases} = \begin{cases} U \exp[i(\beta_1 x + \beta_2 y - \omega t)] \\ V \exp[i(\beta_1 x + \beta_2 y - \omega t)] \\ W \exp[i(\beta_1 x + \beta_2 y - \omega t)] \end{cases} \quad (44)$$

Table 1 Material properties of Terfenol-D

$E_c$ (GPa)	$\nu_c$	$\rho_c$ (kg/m <sup>3</sup> )	$\alpha_c$ (1/K)	$e_{31} = e_{32}$ (N/mA)
30	0.25	9250	12e-6	442.55

Table 2 Material properties of Alumina (Al<sub>2</sub>O<sub>3</sub>)

$E_f$ (GPa)	$\nu_f$	$\rho_f$ (kg/m <sup>3</sup> )	$\alpha_f$ (1/K)	$\beta_f$ (wt. % H <sub>2</sub> O) <sup>-1</sup>
380	0.3	3800	7e-6	0.0001

where  $U$ ,  $V$  and  $W$  are wave amplitudes,  $\beta_1$  and  $\beta_2$  are wave numbers in  $x$  and  $y$  directions, respectively; and  $\omega$  is the circular frequency of dispersed waves. Substituting for  $u$ ,  $v$  and  $w$  from Eq. (44) in the Eqs. (40)-(42), the following equation is obtained

$$([K] + i\omega[C] - \omega^2[M]) \begin{bmatrix} U \\ V \\ W \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \quad (45)$$

Solving the above equation results in finding the circular frequency. Once this value is divided by wave number, the phase velocity can be reached as

$$c_p = \frac{\omega}{\beta} \quad (46)$$

## 3. Results and discussion

In this part, the wave dispersion behaviors of a MSNP are going to be reviewed numerically in order to understand the effect of various parameters. In the present article, central core is presumed to be made of Terfenol-D and the top and bottom face sheets are supposed to be consisted of Alumina. The properties of constituent materials are presented in Tables 1 and 2. Moreover, in Table 3 results of verification are presented which show the accuracy and efficiency of present approach.

In Fig. 2, variation of wave frequency versus wave number for various nonlocal and length scale parameters is depicted to put emphasize on the crucial impact of these coefficients on the dispersion responses of MSNPs. It can be figured out that the wave frequency amounts can be decreased whenever nonlocality is increased from zero to a desirable nonzero value in the case of NE ( $\lambda = 0$ ). Therefore, the stiffness-softening influence can be easily found in this diagram which is the reduction of wave frequency while nonlocal parameter is added. On the other hand, in a certain nonzero amount of nonlocal parameter

Table 3 Comparison of first dimensionless fundamental frequencies of functionally graded nanoplates with respect to nonlocality, plate's thickness and aspect ratio ( $n = 5$ ,  $a = 10$  nm)

$a/b$	$\mu$ (nm <sup>2</sup> )	$a/h = 10$		$a/h = 20$	
		Present	Natarajan <i>et al.</i> (2012)	Present	Natarajan <i>et al.</i> (2012)
1	0	0.0461	0.0441	0.0116	0.0113
	1	0.0422	0.0403	0.0106	0.0103
	2	0.0391	0.0374	0.0098	0.0096
	4	0.0345	0.0330	0.0086	0.0085
2	0	0.1137	0.1055	0.0288	0.0279
	1	0.0930	0.0863	0.0235	0.0229
	2	0.0806	0.0748	0.0204	0.0198
	4	0.0659	0.0612	0.0167	0.0162

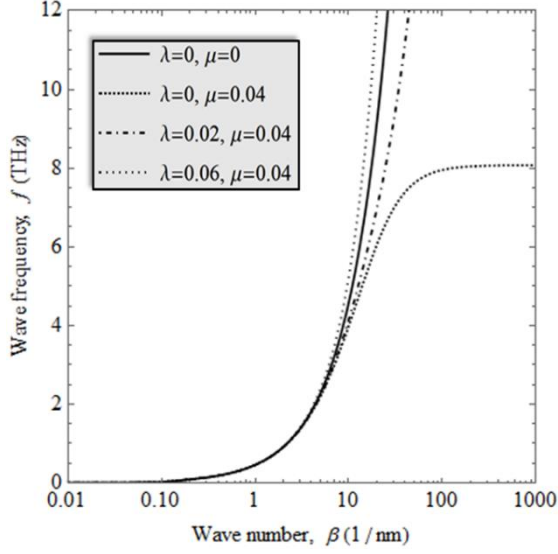


Fig. 2 Variation of wave frequency of MSNPs for different amounts of nonlocal and length scale parameters ( $k_w = 0.01$ ,  $k_p = 0.05$ ,  $c_d = 0$ ,  $\Delta T = 100$ ,  $\Delta C = 0$ )

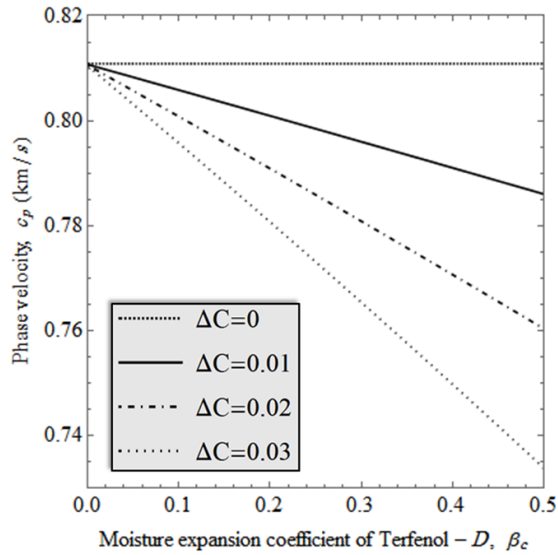


Fig. 3 Influence of moisture expansion coefficient of magnetostrictive core on the phase velocity of MSNPs for various moisture concentration values ( $k_w = 0.01$ ,  $k_p = 0.05$ ,  $c_d = 0.1$ ,  $\Delta T = 100$ ,  $\beta = 0.05$  (1/nm),  $\lambda = \mu$ )

( $\mu = \text{constant}$ ), a rise in the magnitude of wave frequency can be reported as length scale parameter is supposed to be intensified. Thus, length scale parameter is able to aggrandize the value of dispersion answers of MSNPs once it is motivated increasingly. The reason of this phenomenon is exactly the hardening effect of the length scale term on the mechanical responses of nanostructures. Afterwards, Fig. 3 is mainly rendered for the purpose of showing the effect of moisture expansion coefficient of Terfenol-D on the phase velocity values of MSNPs whenever moisture concentration is assumed to be varied. In fact, there is a lack of information about the moisture expansion coefficient of

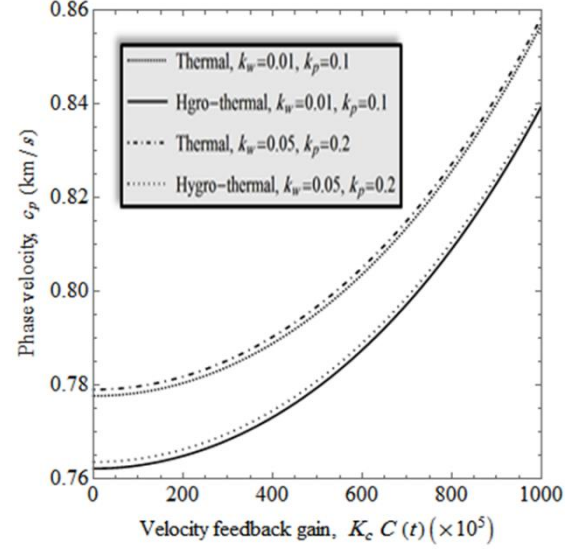


Fig. 4 Effect of velocity feedback gain on the phase velocity of MSNPs under various hygro-thermal loadings and medium conditions ( $\beta = 0.03$ ,  $\lambda = \mu$ ,  $c_d = 0.2$ )

this smart material. However, it is clear that Terfenol-D is consisted of Terbium, Iron and Dysprosium which are metallic materials. Hence, moisture expansion coefficient of Terfenol-D should be in the routine range of this coefficient for metals. In this figure, this coefficient is considered to be varied from 0 to 0.5 and the influences of gradual changes in this coefficient on the wave dispersion speed are observed. Based on the diagram, a linear impact with negative slope can be seen for moisture expansion coefficient whenever a nonzero amount is employed for moisture concentration ( $\Delta C \neq 0$ ). In other words, in the cases of moist environments phase velocity decreases continuously as moisture expansion coefficient of Terfenol-D increases, whereas, it remains constant in dry situations. From now, in all of the future diagrams, the desirable value of 0.3 is aligned to the moisture expansion coefficient of Terfenol-D ( $\beta_c = 0.3$  (wt. %  $\text{H}_2\text{O}$ )<sup>-1</sup>).

Furthermore, in Fig. 4 influence of velocity feedback gain on the phase velocity of MSNPs is taken into consideration for various environments and different values of foundation parameters. Clearly, it can be observed that phase velocity can be controlled by changing velocity feedback gain amounts. Indeed, once this control gain is added, phase velocity of MSNPs becomes greater. On the other hand, it can be perceived that the wave speed grows in the cases of higher Winkler and Pasternak foundations. Also, it is of interest that phase velocity values are bigger under thermal loading compared to the condition in which the structure is placed in a hygro-thermal environment. So, the decreasing effect of moisture concentration can be easily found again in this figure as same as Fig. 3.

Fig. 5 is plotted to show the effects of various environments on the phase velocity of MSNPs with respect to the medium's damping coefficient for various wave numbers. First of all, it is clear that wave dispersion responses go through a continuous decreasing path as damping coefficient increases; however, value of critical damping coefficient which results in the zero value for



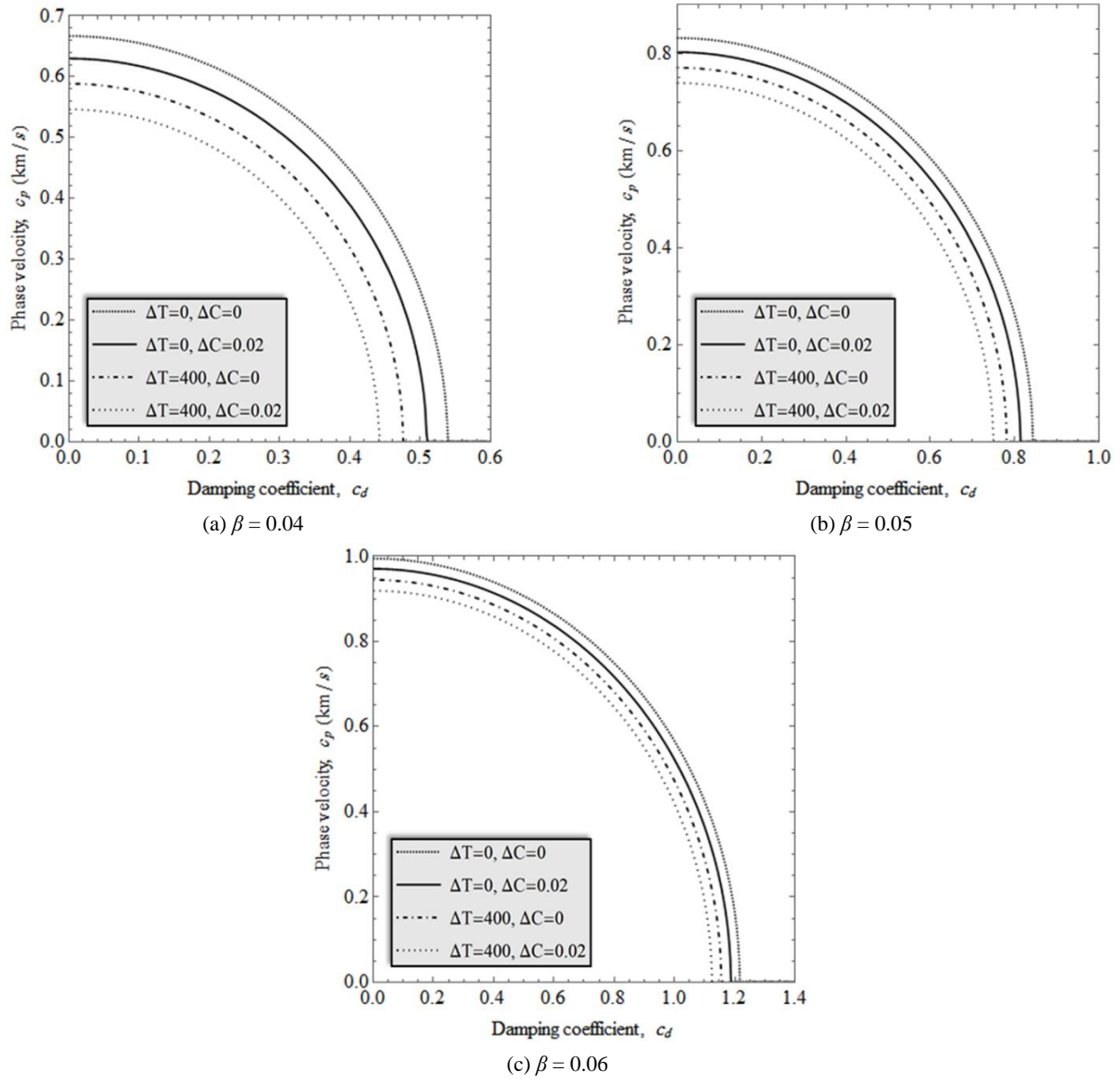


Fig. 5 Variation of phase velocity versus damping coefficient for various temperature rise and moisture concentration values ( $k_w = 0.01$ ,  $k_p = 0.05$ ,  $\lambda = \mu$ )

phase velocity is not constant for all wave numbers. In other words, greater damping coefficients are needed to totally damp the phase velocity of dispersed waves whenever a high wave number is employed. Therefore, in the design procedures, dampers with higher viscosities shall be utilized if high wave numbers are applied. On the other hand, based on the figure, any increase in either temperature rise or moisture concentration leads to smaller wave velocities. Thus, smaller damping coefficients can be used once the system is placed in hygro-thermal and thermal environments compared to the condition of non-hygro-thermal environments.

At last, coupled effects of moisture concentration, temperature rise and foundation parameters are investigated in Fig. 6 drawing the variations of phase velocity versus moisture concentration. Herein, it is obvious that phase velocity values become greater once higher foundation parameters are chosen. Moreover, it can be understood that

a rise in the moisture concentration results in a decrease in the value of phase velocity. Also, a same impact can be reported for probable changes in the temperature rise; means, the maximum amount of phase velocity is a desired moisture concentration can be reached in the case of  $\Delta T = 0$ . So, in the cases that there is a limitation for moisture concentration, higher temperature rises can be utilized instead.

#### 4. Conclusions

Present paper is an analytical method for wave propagation analysis of MSNPs in hygro-thermal environments. In this model, a NSG based plate model is introduced to probe the size effects in details. The magnetostriction effects are covered on the basis of a feedback control system. Nanoplate is rested on a



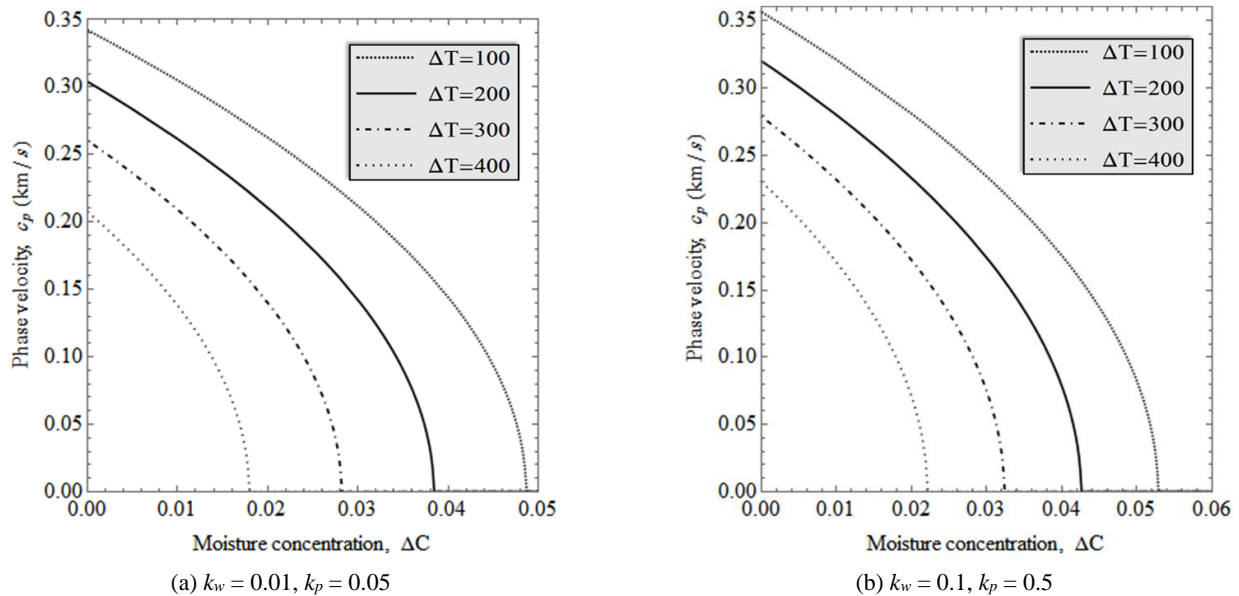


Fig. 6 Variation of phase velocity versus moisture concentration for various amounts of temperature rise and foundation parameters ( $\beta = 0.04$ ,  $\lambda = \mu$ ,  $c_d = 0.2$ )

viscoelastic substrate and modeled by a CPT. On the other hand, governing equations are developed according to the Hamilton's principle. Herein, a summary of most significant results is presented as follows:

- Wave dispersion answers of MSNPs can be amplified by either increasing the length scale parameter or decreasing the nonlocal parameter.
- A rise in the amount of velocity feedback gain results in higher phase velocities.
- Utilizing greater Winkler or Pasternak coefficients, phase velocity values can be added.
- Phase velocity of the system decreases once temperature rise or moisture concentration values are amplified or a higher damping coefficient is used.

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