

Analytical analysis for the forced vibration of CNT surrounding elastic medium including thermal effect using nonlocal Euler-Bernoulli theory

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Abstract. This article studies the free and forced vibrations of the carbon nanotubes CNTs embedded in an elastic medium including thermal and dynamic load effects based on nonlocal Euler-Bernoulli beam. A Winkler type elastic foundation is employed to model the interaction of carbon nanotube and the surrounding elastic medium. Influence of all parameters such as nonlocal small-scale effects, high temperature change, Winkler modulus parameter, vibration mode and aspect ratio of short carbon nanotubes on the vibration frequency are analyzed and discussed. The non-local Euler-Bernoulli beam model predicts lower resonance frequencies. The research work reveals the significance of the small-scale coefficient, the vibrational mode number, the elastic medium and the temperature change on the non-dimensional natural frequency.

Keywords: nanostructures; Winkler's moduls; forced vibration; carbon nanotubes; non-local Euler beam theory

1. Introduction

Carbon nanotube CNTs are cylindrical macromolecules composed of carbon atoms in a periodic hexagonal arrangement discovered by Iijima (1991), which have received tremendous attention from various branches of science. Varieties of experimental, theoretical, and computer simulation approaches indicate that carbon nanotubes (CNTs) possess superior electronic, thermal and mechanical properties (Heireche *et al.* 2009). Other studies have showed that they have good properties so they can be used for nanocomposites. In addition, CNTs are well known for their excellent rigidity, higher than that of steel and any other metal. Zidour (Zidour *et al.* 2014) analysed the Buckling of chiral single-walled carbon nanotubes by using the nonlocal Timoshenko beam theory.

Due to difficulties encountered in experimental methods to predict the responses of nanostructures under different loading conditions, the molecular dynamics (MD) simulations and the continuum mechanics methods are used. But the computational problem when using the (MD) is that the time steps involved in the (MD) simulations are limited by the vibration modes of the

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atoms to be of the order of femto-seconds (10-15 s). The continuum mechanics methods have been effectively used to study mechanical behaviors of not only single-walled carbon nanotubes (SWCNTs) but also MWCNTs (Benferhat *et al.* 2016, Zidour *et al.* 2014, Ben Henni *et al.* 2018, Bouakaz *et al.* 2014, Rabahi *et al.* 2016, Daouadji and Adim 2016, Bensattalah *et al.* 2018a, Aissani *et al.* 2015, Abdelhak *et al.* 2016 and Bensattalah *et al.* 2016). Recently, the continuum mechanics approach has been widely and successfully used to study the responses of nanostructures, such as the static, the buckling, free vibration (Zidour *et al.* 2012, Daouadji *et al.* 2016b, Adim *et al.* 2016a, b, Daouadji 2017, Rabahi *et al.* 2016, Chaht *et al.* 2015, Khelifa *et al.* 2018, Tlidji *et al.* 2014 and Bensattalah *et al.* 2018b), wave propagation (Naceri *et al.* 2011) and thermo-mechanical analysis of (CNTs) (Daouadji and Hadji 2015, Benferhat *et al.* 2016, Hadji *et al.* 2015, Benferhat *et al.* 2016 and Abderezak *et al.* 2018).

At nanoscale, the classical continuum theories are deemed to fail, because the length dimensions at this scale are often sufficiently small such that call the applicability of classical continuum theories into the question. At macroscopic scale, the mechanical characteristics of structures are often significantly different from their behavior at nanoscale. Consequently, many non-local theories that consider the scale effect have been proposed such as deformation gradient theory, micro-polar theory and the nonlocal theory of elasticity (Eringen 1972b), this theories take into account the influence of the screen introducing the intrinsic scale length in the constituent relations. Among the theories mentioned previously, the non-local elasticity theory (Eringen 1983) is considered as a function of strain state of all points in the body.

In the past fifty years, linear and nonlinear problems which appeared in physical, chemistry, mechanics, engineering applications and various of scientific areas are modelled and they are investigated by using so many approximating methods. Some of these numerical methods are Differential Transformation Method (DTM), Homotopy Perturbation Method (HPM), Adomian Decomposition Method (ADM), Variational Iteration Method (VIM) and Homotopy Analysis Method (HAM). Many authors studied linear and nonlinear models to compute approximate solutions and their convergences with Differential Transformation Method (DTM) (Hassan 2002, Sallai *et al.* 2015, Hadji *et al.* 2016a, b, Heireche *et al.* 2008, Ju 2004).

Recently, three-dimensional behaviour of CNT is investigated by some researchers (Yuzhou and Liew 2008, Gupta and Batra 2008). CNT can be used for micro-electro-mechanical and nano-electro-mechanical devices. During these applications, external forces act on CNTs continuously. Because of this fact, understanding their dynamic behavior is a very important task. Forced vibration of CNT-reinforced epoxy is studied by Rajoria (Rajoria and Jalili 2005). According to the authors' best knowledge, forced vibration of CNTs by using continuum beam models was not studied in the previous research. For this reason, we presented in this article studies the free and forced vibrations of the carbon nanotubes (CNTs) embedded in an elastic medium including thermal and dynamic load effects have been extracted via the theory of nonlocal continuum elasticity. The mathematical derivations and numerical investigations are presented and performed while the emphasis is placed on investigating the impact of all different parameters such as nonlocal small-scale effects, high temperature change, Winkler modulus parameter, vibration mode and dynamic load. Comparisons of present approach with the results from the existing literature are provided and the good agreement between the results of the proposed method and those available in literature validated the presented approach.

2. Nonlocal Euler-Bernoulli elastic beam models

The theory of nonlocal continuum elasticity proposed by Eringen (1983) assumed that the stress at a reference point is considered to be a functional of the strain field at every point in the body. In the limit when the effects of strains at points other than x are neglected, one obtains local or classical theory of elasticity. For homogeneous and isotropic elastic solids, the constitutive equation of non-local elasticity can be given by Eringen (1972a). Non-local stress tensor (t) at point (x') is defined by

$$\begin{aligned} \sigma_{ij,j} &= 0 \\ \sigma_{ij}(x) &= \int K(|x-x'|, \tau) C_{ijkl} \varepsilon_{kl}(x') dV(x'), \quad \forall x \in V \\ \varepsilon_{ij} &= \frac{1}{2}(u_{i,j} + u_{j,i}) \end{aligned} \tag{1}$$

where C_{ijkl} is the classical, macroscopic stress tensor at point x' , σ_{ij} and ε_{ij} are stress and strain tensors respectively. $K(|x-x'|, \tau)$ is the kernel function and ($\tau = e_0 a/l$) is a material constant that depends on internal and external characteristic length (such as the lattice spacing and wavelength), where e_0 is a constant appropriate to each material, a is an internal characteristic length, e.g., length of (C-C) bond, lattice parameter, granular distance, and (l) is an external characteristic length.

Non-local constitutive relations for present nano-beams can be approximated to a one-dimensional form as

$$\left(1 - e_0 a^2 \frac{\partial^2}{\partial x^2}\right) \sigma_x = E \varepsilon_x \tag{2}$$

where E is the Young's modulus and the scale coefficient ($e_0 a$) in the modelling will lead to small-scale effect on the response of structures at nano-size. Assume that the displacement of the beam along the z axis is $w(x, t)$ in terms of spatial coordinate x and time variable t (Fig. 1).

For transverse vibration of nanotube, the equilibrium conditions of the Euler-Bernoulli beam can be written as

$$\frac{\partial V}{\partial x} = \rho A \frac{\partial^2 w}{\partial t^2} - P(x) - F(x, t) \tag{3a}$$

$$V = \frac{\partial M}{\partial x} - N \frac{\partial w}{\partial x} \tag{3b}$$

where V and M are resultant shear force and bending moment of the beam, ρ is the mass density, A is the area of the cross-section of the beam, w is the transverse displacement of the micro-tubules, $f(x, t)$ is the distributed transverse force along the x -axis, $P(x)$ is the inter action pressure per unit axial length between the nanotube and the surrounding elastic medium, and t is the time variable. N is the axial force arising from the thermal effect and it is defined as

$$N = -EA\alpha\theta \tag{4}$$

In addition the pressure per unit axial length, acting on the outermost tube due to the

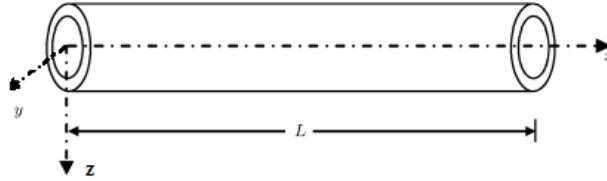


Fig. 1 The illustration of carbon nanotube

surrounding elastic medium, can be described by a Winkler type model.

$$P(x) = -K_{win} w(x) \quad (5)$$

Where the negative sign indicates that the pressure $P(x)$ is opposite to the deflection of the outermost tube and K_{win} is spring constant of the surrounding elastic medium.

The elasto-dynamics differential equation that governs the thermo-mechanical vibration of the nanotube SWCNT based on the nonlocal Euler-Bernoulli beam theory is

$$EI \frac{\partial^4 w}{\partial x^4} + \left(1 - e_0 a^2 \frac{\partial^2}{\partial x^2}\right) \left[\rho A \frac{\partial^2 w}{\partial t^2} - N \frac{\partial^2 w}{\partial x^2} - k_{win} w \right] = \left(f - e_0 a^2 \frac{\partial^2}{\partial x^2} f \right) \quad (6)$$

Since finding an analytical solution is possible for simply supported boundary conditions for the present problem, the (SWNT) beam is assumed simply supported. For the forced flexural vibrations, the applied dynamic load $f = f(x, t)$ is assumed as a result, the boundary conditions have the following form

$$w(x, t) = \sum_{m=1}^{\infty} W_m \cdot \sin\left(\frac{m\pi x}{L}\right) \cdot \sin(\omega t) \quad (7a)$$

$$f(x, t) = \sum_{m=1}^{\infty} F_m \cdot \sin\left(\frac{m\pi x}{L}\right) \cdot \sin(\omega t) \quad (7b)$$

Here ω is the operational frequency of the load. Writing in Eq. (6), setting $e_0 a = 0$, $N = 0$ and $K_{win} = 0$ give equation of motion for forced vibration of CNTs in the local form

$$EI \frac{\partial^4 w}{\partial x^4} + \left[\rho A \frac{\partial^2 w}{\partial t^2} \right] = f \quad (8a)$$

The simply supported boundary conditions can be written as

$$w = M = 0 \quad \text{at} \quad x = 0, L \quad (8b)$$

Following displacement field satisfies equation of motion (equation 6) and boundary conditions (Eq. (8)). Inserting Eq. (7) into Eq. (6) and solving W_m gives the amplitude of CNT in the following form

$$\lambda_{LE} = \left(\frac{F_0 L^4}{EI} + \beta^4 \right)^{0.25} \tag{9}$$

where $\beta = m\pi$, without omitting all parameters e_0a , N and k in Eq. (6) and doing similar operations mentioned above, the following amplitude can be found for the nonlocal case

$$\lambda_{NE} = \left[\frac{F_0 L^4 (1 - \xi)}{EI(1 + \xi)} + \frac{(\beta^4 - \beta^2 \bar{N}(1 + \xi) + \mu(1 - \xi))}{(1 + \xi)} \right]^{0.25} \tag{10}$$

The non-dimensional parameters for the Euler-beam on the Winkler foundation are defined as

$$\xi = \frac{\beta^2 e_0 a^2}{L^2}, \quad \lambda^4 = \frac{\rho A \omega^2 L^4}{EI}, \quad \mu = \frac{k L^4}{EI}, \quad \bar{N} = \frac{N \cdot L^2}{EI}, \quad \varepsilon = \frac{e_0 a}{L} \tag{11}$$

The Young’s modulus, thermal expansion coefficients of CNTs and the spring constant of polymer matrix, under temperature changes environments, which may be a function of temperature change as follows

$$\begin{aligned} E &= E^0(1 - 0.0005\theta) \\ N &= -EA\alpha\theta = \alpha^0(1 + 0.002\theta) \\ k &= k^0(1 - 0.0003\theta) \end{aligned} \tag{12}$$

where E^0 and α^0 express elastic modulus and thermal expansion coefficients of CNTs under a room temperature environment, respectively. k^0 is spring constant of polymer matrix under a room temperature environment.

3. Results and discussion

In the present study the impact of all parameters such as nonlocal small-scale effects, high temperature change, Winkler modulus parameter, vibration mode for four types of boundary conditions e.g. simply supported, clamped- simply, clamped ends and Cantilever beam on first,

Table 1 First three non-dimensional frequency λ of nonlocal Euler-Bernoulli for simply supported beam, with ($k = 0, \theta = 0, F_0 = 0, L/d = 10$)

r	Mode 1		Mode 2		Mode 3	
	Wang 2007	Present	Wang 2007	Present	Wang 2007	Present
0	3.1416	3.141593	6.2832	6.283185	9.4248	9.424777
0,1	3.0685	3.068531	5.7817	5.781668	8.0400	8.039987
0,3	2.6800	2.679996	4.3013	4.301343	5.4422	5.442246
0,5	2.3022	2.302231	3.4604	3.460401	4.2941	4.294061
0,7	2.0212	2.021245	2.9585	2.958479	3.6485	3.648549

second and third frequencies of the SWCNTs have been studied.

The material properties used in the present study are the mass density $\rho = 2300 \text{ kg/m}^3$, the Poisson ratio $\nu = 0.19$, the Young's modulus $E = 5.5 \text{ MPa}$ and the thermal expansion coefficients at high temperature $\alpha_0 = 1.1 \times 10^{-6} \text{ K}^{-1}$ (Yao and Han 2006). Another, Young's modulus, thermal expansion coefficients of CNTs and the spring constant of polymer matrix, under temperature changes environments, which may be a function of temperature change as present in Eq. (12). In the beginning of this study, the validation our model local Euler-Bernoulli for simply supported beam boundary condition is presented in Table 5, with case $\alpha = 0$, $\mu = 0$, $\theta = 0$, $k = 0$, $F_0 = 0$ and $L/d = 10$. The temperature effect and Winkler modulus parameter have been considered here. The details of first three non-dimensional frequency λ for four kinds of boundary conditions with and without elastic medium using nonlocal Euler-Bernoulli beam model are listed in table 2. The ratio of the length to the diameter, (L/d), is 10 and the scale coefficients ($\varepsilon = 0, 0.5, 1$).

The results show the dependence of the frequency on the mode number and the elastic medium. It is noted that the frequency increases when elastic medium is neglected. This increasing is attributed to the stiffness of the elastic medium. With higher values of mode number the rate of increase of frequency reduces, and becomes more significant with the higher of small-scale parameter. This is interpreted as the small-scale effect makes the CNTs more flexible as CNT being assumed as atoms linked by springs, the external elastic medium "grips" the SWCNTs and forces it to be stiffer. In addition, it is clearly that the frequency increases when the vibrational mode number increases.

The results in Table 3 show the dependence of the frequency of first three non-dimensional frequency λ with temperature change using nonlocal Euler-Bernoulli beam model. The ratio of the length to the diameter, (L/d), is 10 and the scale coefficients, ($\varepsilon = 0, 0.5, 1$). We consider the temperature change values as $\theta = 20, 40$ and 60 .

It is noted that the first three non-dimensional frequency λ decreases as the temperature change θ increases, and becomes more significant with the small-scale parameter. This dependence of the frequency on the temperature change appraises in simply supported beam.

Table 2 The effect of Winkler modulus parameter on first three non-dimensional frequency λ of nonlocal Euler-Bernoulli for simply supported beam and ($L/d = 10$, $\theta = 0$, $F_0 = 0$)

ε	Without elastic medium			With elastic medium		
	Mode 1	Mode 2	Mode 3	Mode 1	Mode 2	Mode 3
0	3.141593	6.283185	9.424778	3.102895	6.278431	9.423370
0.5	2.302231	3.460401	4.294057	2.198958	3.431617	4.279104
1	1.730201	2.491001	3.061400	1.435812	2.411026	3.019494

Table 3 The effect of temperature change on First three non-dimensional frequency λ of nonlocal Euler-Bernoulli for simply supported beam with ($k = 0.1$, $F_0 = 0$ and $L/d = 10$)

ε	$\theta = 20$			$\theta = 40$			$\theta = 60$		
	Mode 1	Mode 2	Mode 3	Mode 1	Mode 2	Mode 3	Mode 1	Mode 2	Mode 3
0	3.140149	6.282464	9.424297	3.138592	6.281686	9.423779	3.136921	6.280853	9.423224
0.5	2.298556	3.456074	4.288967	2.294578	3.451396	4.283459	2.290293	3.446363	4.277537
1	1.721497	2.479342	3.047268	1.711975	2.466599	3.031828	1.701592	2.452723	3.015015

Table 4 The effect of dynamic load on First three non-dimensional frequency λ of nonlocal Euler-Bernoulli for simply supported beam With ($k = 0.1$ end $L/d = 10$)

ε	$F_0 = 10^{-3}$			$F_0 = 10^{-2}$			$F_0 = 10^{-1}$		
	Mode 1	Mode 2	Mode 3	Mode 1	Mode 2	Mode 3	Mode 1	Mode 2	Mode 3
0	3.181725	6.289413	9.427180	3.178361	6.288978	9.427050	3.144121	6.284625	9.425759
0.5	2.268336	3.446134	4.2910530	2.272248	3.448288	4.292303	2.310303	3.469618	4.304749
1	1.5311616	2.4412869	3.0504308	1.555180	2.448323	3.054150	1.75006	2.515568	3.090617

In Table 4 show the dependence of the frequency of first three non-dimensional frequency λ with dynamic load change using nonlocal Euler-Bernoulli beam model. The ratio of the length to the diameter, $L/d = 10$, and the scale coefficients, ($\varepsilon = 0, 0.5, 1$). We consider the temperature change values as $F_0 = 10^{-3}$ nN, $F_0 = 10^{-2}$ nN, $F_0 = 10^{-1}$ nN. It is noted that the first three non-dimensional frequency λ decreases as the dynamic load change F_0 increases, and becomes more significant with the small-scale parameter. This dependence of the frequency on the temperature change appraises in simply supported beam.

The results are including and excluding, in order to study the effect of the small-scale parameter, the elastic medium and the change of temperature on the vibrations of single-walled nanotubes.

The Fig. 2 shows the dependence of the frequency ratios on the small scale and vibrational mode number of (SWCNTs) embedded in an elastic medium using thermal effects. The frequency ratio serves as an index to assess quantitatively the scale effect on CNT vibration solution. It is observed from Fig. 2 that the frequency ratios are less than unity. This means that the application of the local Euler-Bernoulli beam model for CNT analysis would lead to an over prediction of the frequency if the scale effect between the individual carbon atoms in CNTs is neglected. However for larger values of α , this dependence becomes very largest. However the small scale effect makes the beam more flexible. In addition, it is clearly that as the vibrational mode number increases, this dependence becomes very largest. This significance in higher modes is attributed to the influence of small wavelength for higher modes. For smaller wavelengths, interactions between atoms are

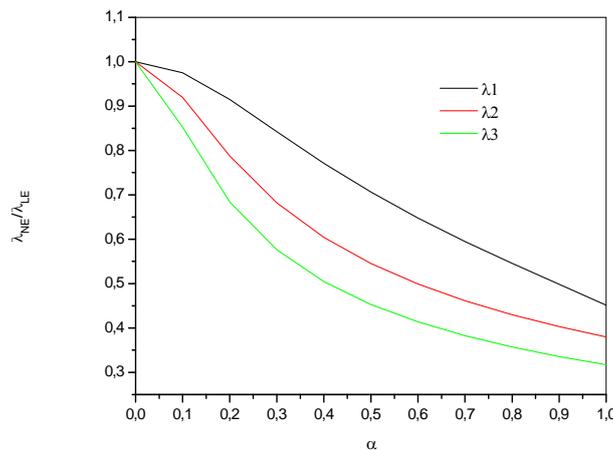


Fig. 2 Small scale effect on different frequency modes for simply supported beam and ($k = 0.1, \theta = 40, L/d = 10$)

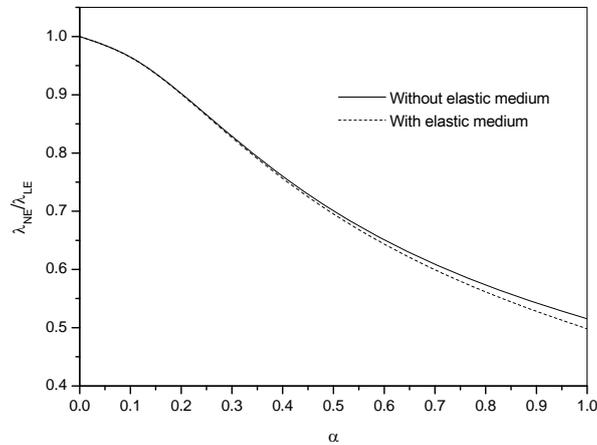


Fig. 3 The effect of elastic medium on first non-dimensional frequency of a short-SWCNT for simply supported beam with different parameter α and ($\theta = 40, L/d = 10$)

increasing and this leads to an increasing in the small scale effect. The effect of elastic medium on first and third non-dimensional frequency of a short-SWCNT for simply supported beam with different parameter α is shown in Fig. 3 with the aspect ratio is $L/d = 10$ and $\theta = 40$. It can be seen that the difference between the frequency ratios with and without elastic medium is very weak for small values of α and for the higher values this difference become clear. In addition, the range of the frequency ratios without elastic medium is the smallest for frequency ratios with elastic medium because the elastic medium grips the CNT and forces it to be stiffer.

The Fig. 4 illustrates dependence of the first and third non-dimensional frequency ratios on the temperature change with the aspect ratio $L/d = 10$ and with elastic medium. It is clearly seen from Fig. 4 that the frequency ratios are less than unity and the difference between the frequency ratios

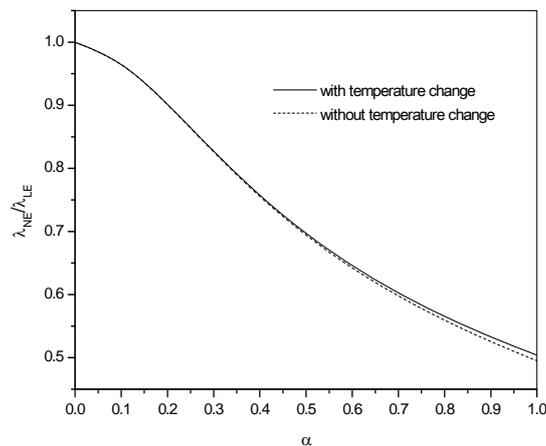


Fig. 4 The effect of temperature on first non-dimensional frequency of a short-SWCNT for simply supported beam with different parameter α and ($k_{win} = 0.1, L/d = 10$)

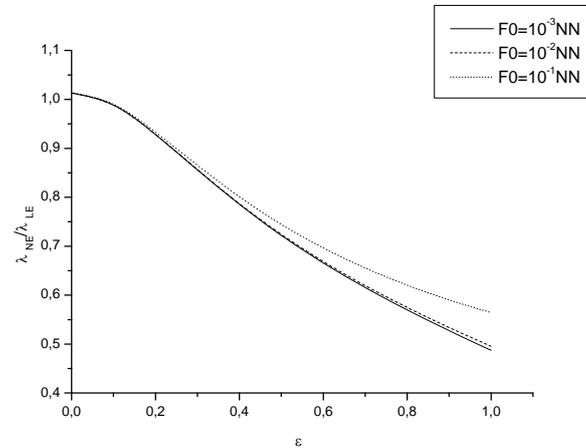


Fig. 5 The effect of dynamic load on non-dimensional frequency of a short-SWCNT for clamped ends with different parameter α and ($m = 1, k = 0.1, L/d = 10$)

with and without temperature change is clear for higher values of α and in first mode.

The Fig. 5 illustrates influence of dynamic load of the first and third non-dimensional frequency ratios with the aspect ratio is $L/d = 10$ and with elastic medium. It is clearly seen from Fig. 5 that the frequency ratios are less than unity and the difference between the frequency ratios at temperature 40 is clear augment if dynamic load augment for higher values of ε and in first mode.

4. Conclusions

Based on local and non-local Euler–Bernoulli beam theory, free and forced vibration of CNT embedded in an elastic medium including thermal and dynamic load effects is studied. Non-dimensional frequency ratios for the local and non-local Euler beam models are given for SWCNTs and DWCNTs. It is found that the non-local models give higher amplitudes when compared with the local Euler–Bernoulli beam models. This study can be extended to other boundary conditions and Timoshenko beam theory. The presented investigation may be helpful in the application of (CNTs), such as nano-electronics, nano-devices, mechanical sensors and nano-composites.

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